

TEST

OF

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MALLET

ARTICULATED

COMPOUND

LOCOMOTIVE

ON

ERIE

RAILROAD

BY

CHARLES R. CULLEN

&

SIDNEY DIAS GRIDLEY

1908.

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Erie Mallett Articulated Compound Locomotive.

Preface.

The authors wish to extend their thanks to the Erie Railroad for its kind loan of locomotive 2602 for the test, and for materials and aid in fitting up the locomotive. We are especially indebted to the following officials of the Erie:-
Mr. A.G.Trumbull, Mechanical Superintendent;
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all obstacles, and bringing the test to a successful
conclusion.

We have made use of all available material
in our descriptions, etc.. The history of the compound
locomotive is taken from Gairus' "Locomotive Com-
pounding and Superheating" and from a thesis pub-
lished in 1891 by E.C.Lombard and H.G.Van Everen.
The figures concerning the B.&O. locomotive are
taken from a paper used by Mr. J.E.Muhlfield of that
road before the New York Railway Club; while the
description of the Erie locomotive is taken from
"The Railway Age", "The American Engineer and
Railroad Journal", and from prints and information
furnished by the American Locomotive Co. The
results of the tests of the Pennsylvania Railroad
at the St. Louis Exposition were also consulted
frequently.

Introduction.

The introduction of the Mallet articulated compound locomotive in America has been viewed with intense interest by all railway men. These locomotives with their immense power, coupled with ability to get around sharp curves, and their very favorable distribution of weight, allowing great increase in the weight of the locomotive without undue strain on the track and road bed, have given us a new and remarkably efficient factor in freight transportation. Their ability to haul the same trains with the same total motor weight without increasing the strain on the rails and under more favorable conditions of adhesion, since the total weight of the locomotive is on the drivers; their offering of less resistance on sharp curves than the large consolidation freight locomotives, in use to day, due to their shorter rigid wheel-base; and their saving in fuel and water by the use of compounding have made them a factor to be reckoned with by every railroad official.

These locomotives are at present mostly used as pushers up heavy grades where the freight traffic is very heavy. The Erie Railroad having put three of these locomotives in service and used them for six months, it was desired to give them a general test as to power developed, efficiency, and fuel

and water consumption. The Erie, having granted their consent to this test, the results obtained are here inclosed.

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HISTORY.

The first attempt at compounding locomotives was made by Roentgen in France in 1834. This however was not developed practically. In 1850 Nicholson and Samuels patented a continuous expansion engine, which was used on the Eastern Counties Railway, now the Great Eastern Railway of England. Steam was admitted to the one cylinder for half the stroke; the communication with the other cylinder, which was on dead-center, was established, and the steam expanded in both cylinders, thus allowing expansion in the first cylinder for half stroke and in the second cylinder for full stroke. These engines were not strictly compound, and the saving in cylinder condensation sought by compounding was not attained.

In 1853 Sutcliffe invented a three-cylinder arrangement, with two high pressure cylinders outside operating the driving axle, and a low pressure cylinder inside, on an axle geared to the driving axle by a two to one gear. This cylinder ran at double the speed of the two high pressure cylinders, and was supplied alternately by their exhaust. In 1859 Salmon proposed a four cylinder engine, with two high pressure cylinders at the fire-box end,

driving the front axles, and two low pressure cylinders at the smoke-box end, driving the rear axles ; the connecting rods crossed, being in different planes.

In 1860 a tandem compound was invented by Kemp , and in 1866 Joy patented a compound in which the high pressure cylinders drove small drivers the low pressure cylinders larger ones , the different speeds compensating for the difference in pressure. Several modifications of the preceding systems were suggested during the next few years , but none proved practical.

In 1874 Monsieur Anatole Mallet patented a compound system in France ; this is one of the two systems of compounding in use today , the other being the Wordell - von Borries . In the Mallet system the engineer is given control of the use of boiler steam in the low pressure cylinders, while in the Wordell - von Borries system the engine is automatically converted into a compound after starting simple . This control is established by means of a distributing valve admitting live steam to both cylinders independently, while allowing the exhaust from the high pressure cylinder to pass into the stack. At first no reducing valve was used for the low pressure cylinders , and when working simple these cylinders did about

two or three times as much work as the high pressure cylinders . Later Mallet designed a reducing valve to obviate this , by means of which the live steam is admitted to a chamber through a small orifice allowing expansion and consequent reduction of pressure .

The term articulated, as applied to locomotives , means, that the locomotive is so constructed that some of the driving wheels may move in planes at an angle to the planes of the other driving wheels. This is done by having two sets of driving wheels operated from separate cylinders thus allowing movement around sharp curves, although the wheel base is long. The usual type of articulated locomotive has part of the drivers on a bogie truck, operated by one set of cylinders , the other set of cylinders operating a set of drivers on the frame fastened rigidly to the boiler. Compounding is almost a necessity , as with engines of any power four cylinders are necessary and the boiler could not supply these while acting simple. This most common type of articulated locomotive was introduced by Meyer in 1867 , and adapted for compounding by Mallet in 1884 .

In the Mallet articulated compound locomotive the high pressure cylinders drive the rear or fixed set of drivers , while the low pressure cylinders drive the front drivers on the bogie truck. This

is done for the reason that the necessary ball-and-socket and telescopic steam joints can be made steam tight more easily for low pressure steam , and also because the exhaust is required to pass up the stack at the front end.

The Mallet articulated compound locomotive was first used in 1887 on the Decauville road and later on the St. Gothard Railway , and was quite extensively used in Europe before being introduced into America. The first large locomotive of this type in America was built by the American Locomotive works for the Baltimore and Ohio Rail Road in 1904 ; this locomotive was exhibited at the St. Louis Exposition , and was at that time the largest locomotive in the world. At the close of the exposition , the locomotive was put in service between Connellsville and Rockwood, a mountain division 43.4 miles long , in places the ruling grade being 1%. The locomotive often, singly , moved loads of 2300 tons over this division in four hours . At the end of one year some figures of its performance were made public : Among these:-

Total locomotive milage(computed) 44,976 mi.

Miles run per 2000^lbs, 20 to 40%

volatile coal

9.25 mi.

Gallons of water per 100 miles 15,207 gals.

Pounds of water per lb. coal

1487¹/₂

Miles run per pint engine oil	145 mi.
" " " " valve "	200 mi.
" " " pound crankpin grease	294 mi.
" " " ton of sand	485 mi.

Cost of labor and material for
repairs , per 100 miles \$3.16
Total operating and maintenance expenses,
including all wages of engineers and round
house men per 100 miles \$24.50

The engine lost about 6% of its time in
repairs.

It was estimated that the cost of a general
overhauling , at the end of the year, to put the
locomotive in condition for another years service
would be about \$1000.

The following features of the locomotive
seemed to give general satisfaction :-

- (1). Flexible joints to high and low
pressure cylinder , receiver and exhaust
pipes.
- (2). Articulated frame.
- (3) . Intercepting , reducing and emergency
valve and intermediate chamber system of
compounding and simpling.
- (4). Combination hand and power reversing
gear.
- (5). Walschaert motion valve gear.
- (6). High pressure piston and low pressure
double ported slide valves.

(7) . Single disc main throttle valve.

(8) . Tracking and riding qualities.

Most of these features have been incorporated in the later designs.

This type of locomotive gave the following results :-

(1) Large tractive power for starting trains on heavy grades.

(2) Hauling of trains with the same total motor weight without increasing the strain on the rails, and under more favorable conditions of adhesion, since total weight is on the drivers.

(3) Offering of less resistance on sharp curves.

(4) Saving of fuel by compounding.

(5) Elimination of retarded movement and stalling of trains because of slipping of drivers since, if one set starts to slip, the other takes up more work, and the locomotive grips the track without throttling the steam supply.

(6) A minimum capital investment , repairs, fuel, crew, and supply bill per ton mile.

(7) A sub-division of power and balancing, resulting in minimum strains on locomotive and track.

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(8) Ability to move itself and one half its rating in case one set of machinery or engine becomes disabled.

Because of the success of this locomotive, in 1906 the Great Northern Rail Road ordered five of them for use as pushers on mountain grades, and later ordered twenty-five more for heavy road service.

In 1907, the Erie Rail Road ordered three locomotives of this type for use as pushers on the road between Susquehanna, Pa., and Gulf Summit N.Y., a distance of eight miles, on a ruling grade of 1.3%. At this point the heavy freight traffic of the Erie line between New York and Chicago is carried over the divide between Susquehanna and Delaware Valleys. These engines took the place of three pushers, whereas the B & O locomotive had taken the place of two pushers. These engines were numbered 2601, and 2600, and 2602. This test was run on Number 2602.

A comparison of these Erie locomotives with the B. & O. and the two orders of Great Northern locomotives of this class is given on the next page.

Owner	Erie.	G.N.	G.N.	B.&O.
Number of Locomotives.	3	25	5	1
Type.	0-8-8-0	2-6-6-2-	2-6-6-2	0-6-6-0
Builder.	American Loco. Co.	Baldwin Loco. Co.	Baldwin Loco. Co.	American Loco. Co.
Total Weight in pounds.	409,000	288,000	355,000	334,500
Weight on drivers #.	409,000	250,000	316,000	334,500
Tractive effort, #.	94,8000	57,760	71,600	70,000
Size of cylinders.	25"x39"x28"	20"x31"x30"	21"x33"x32"	20"x32"x32"
Diameter of boiler.	84"	72"	84"	84"
Steam pressure in pounds.	215"	210"	215"	235"
Diameter of drivers.	51"	55"	55"	56"
Total heating surface, sq.ft.	5313.7	3906	5703	5600
Total weight divided by heat.surf.	76.9	73.8	62	59.5
Total heating surface divided by cylinder vol.	222	329	275	295
Total heating surface divided by grate area.	53	73	73	77.3
B.D.Factor.	910	813	690	700

By this it will be seen that in 1908, when these tests are made, these locomotives are the largest and most powerful in existence.

Method.

The test consisted of a series of five runs. The first four runs were a general test, while the fifth run was made merely for coal and water consumption.

Readings were taken every three minutes for steam pressure, injector, reverse lever and throttle positions, pressure of calorimeter in inches of mercury and water, and temperature in branch pipe from steam dome to high pressure cylinder, revolutions per minute on front engine, revolutions per minute on rear engine, cards from each end of each cylinder, temperature and pressure in receiver, temperature of exhaust steam, pyrometer, and vacuum manometer in inches of water. Water was measured at beginning and end of run, coal was weighed as required, and flue gas analysis were made as frequently as possible. In the dynamometer car, besides the record, the mile posts were marked as were the places on the runs where readings were taken. A clock arrangement marked off fifteen second periods on the record.

Signals were given by a short blast on the locomotive whistle. This worked very well as is shown by the fact that every observer's book checked up in number of readings at end of each run. Effort was made to start the test just as the

engine was nearing the iron bridge at Lanesboro.

A preparatory signal for starting test was two short whistles followed in about a minute time by the first short blast. This indicated the first reading of the run.

The Coal.

The coal was weighed up as needed for the fire. This was arranged by having a wooden partition placed in the tender, behind which was the coal. In front was a scales mounted on a platform, and on the scales was a can into which the coal was shoveled; it was then weighed and dumped out for the use of the fireman.

The Water.

The tender was unhitched while the engine was fitting up, and was emptied of all coal and filled to the top of tank with water. This height was measured.

Tender was then placed on scales, previously calibrated, and water was let out an inch at a time, the tender being weighed each time. When all the water was out, the tender was weighed empty, and the other readings minus the weight of the tender gave the weight of water for each inch. Thus when the test was started, a reading was made and when the test was finished a reading was also made, and the weight at the first reading minus weight at the second gave the weight of water used.

The injector was in service nearly all the time, and while running, a leak occurred. This leak was weighed up for the time the injector was on during the run, and a correction subtracted from the total water used to cover the leakage.

The steam pressure was read in the cab from a guage calibrated with a standard guage and found to be accurate. The positions of the reverse lever and throttle were also observed in the cab, this being done by observing the notches from full gear.

The revolutions per minute were recorded for both front and rear engines by means of revolution counters. These counters recorded for each revolution and were so placed that the lever of the counter was connected to the rocker of the Waelschaert valve motion. Readings were taken on these every minute, and time recorded so as to check up at end of run with other observer's time.

For the quality of the high pressure steam a throttling calorimeter was used. This was inserted into the branch pipe about midway between the entrance to the cylinder and the return pipe in the boiler. The manometer read in inches of mercury. Temperature of the steam was also read, as was the steam pressure in the branch pipe. From this data the quality was obtained.

On the receiver a similar arrangement was

placed, just at the point where receiver separated into the two pipes leading to the two low pressure cylinders. The exhaust steam temperature was read on the thermometer placed just at the elbow of the exhaust pipe.

The vacuum guage was arranged so that a pipe perforated on the inner end was inserted on the left side of the smoke box. The perforated end of the pipe was in front of the diaphragm, and on the other end outside of the smoke box, a manometer was adjusted to read in inches of water.

The pyrometer was inserted on the right side of the smoke box, and behind the diaphragm in such a position with respect to the tubes that an average temperature could be obtained; but owing to poor adjustment of the pyrometer itself, the readings can not be relied on to any extent. A calibration of this pyrometer was made and the curve is shown in this report.

For the flue gas samples, a pipe was inserted thru the front plate of the locomotive, reaching to the tubes. By this means a good sample of the flue gas was obtained and the steam jet was avoided.

The flue gas was analyzed immediately after each sample was taken. The apparatus used was the Orsat flue gas case, which consisted of three tubes, one with sodium hydroxide, one with phosphorus pentoxide, and the third with cuprous chloride. The hydroxide mixture was used to extract the CO_2 in the gas,

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the phosphorus to extract the oxygen, and the cuprous chloride to remove the CO. The remainder of the gas was assumed to be Nitrogen.

The indicators were four in number, and were of the Thompson make. The holes in the cylinders were tapped for $\frac{3}{4}$ inch pipe, and were in a plane parallel to the ground. The pipes were given radii of about four inches and were made with easy curves. The piping from each hole to the three way cocks was in two pieces. One piece had a radius of about four inches and was screwed into the tap. The other piece had a long gradual curve, being connected at one end to the first pipe by a coupling and at the other end to the three way cock, so a reducing coupling was necessary. This was fitted for $\frac{1}{2}$ " pipe. The piping was so arranged that the indicator drum would unroll in a straight line parallel to the motion of the cross head. All the piping was heavily covered with asbestos and canvass, and offered very little opportunity for radiation. The reducing motion was accomplished by means of reducing wheels. These wheels allowed an adjustment of stroke that would give a good length of card. Owing to the difficulty of fastening and unfastening the string to the reducing wheel and on the cross head upright, an iron plate was so fastened to each indicator that the reducing wheel was about eight inches from the indicator;

between the indicator and reducing wheel the string could be hooked and unhooked. This did away with the necessity of attaching the card to the upright each time, allowing the string from the upright on the cross head reducing wheel to keep in motion, while the indicator could be stopped for adjusting cards by unfastening hooks on strings between the reducing wheel and the indicator. The diagram for the head and crank end of each cylinder for the same reading was taken on the same card.

Upright.

To get the same motion to the reducing wheel as the crosshead, an iron plate was bolted to the crosshead to which was fastened an upright, $\frac{1}{2} \times \frac{1}{2}$ stock, securely braced. The upright moved in a straight line; as far as the indicator motion was concerned, this did not introduce the error of a varying angle as the crosshead changed its position. Cards were taken on the low pressure cylinder by having the indicator men sit on the cylinders; but on the high pressure cylinders, owing to the lack of room, a platform had to be built in front of the cylinders. This platform was held in place by two iron supports, fastened at the top to the cab floor, and held rigidly at the bottom by a clamp attached to the relief cock of the high pressure cylinder. A flooring on this brace completed the platform. The men could get on and off

at will, owing to the slow speed of the locomotive and were in no danger. A railing protected the men at the front end. This was erected on the front of the engine. This consisted of an iron brace, shaped like an inverted U and fastened to the bypass valve studs on the front and back of the low pressure cylinders, these bracings being connected by a crossbar at the front.

The Dynamometer Car.

This car was the Erie dynamometer car which has a capacity of 70,000 pounds either in tension or compression. It was in charge of the Superintendent of Tests of the Erie R.R. The dynamometer was of the spring type; by means of a system of levers it moved a beam across the paper. The paper was attached to a set of rollers; these were fixed to a mechanism or train of gears, intermeshing with the car axle, causing the paper to move one foot to the mile. To the beam indicating tension or compression a pencil was attached giving a graphical record. To a stationary beam was fixed a pencil, giving the zero or datum line. Lines were drawn on either edge of the paper one recording a mark every fifteen seconds, the other by means of an electrical circuit recording at will, marks for mile posts and readings. The dynamometer car also contained two speed recorders but because of the very slow speed of the engine, these could not be used owing to their inaccuracy.

For the running of the test the following men were necessary:

- 3 men to shovel coal.
- 1 " " weigh coal and take water measurements.
- 1 " in engineer's cab.
- 1 " for high pressure calorimeter.
- 1 " " speed of front engine.
- 1 " " " " rear " .
- 1 " " pyrometer.
- 1 " " vacuum manometer.
- 1 " " low pressure calorimeter and exhaust temperature.
- 1 " " flue gas.
- 1 " " high pressure right indicator.
- 1 " " low pressure right indicator.
- 1 " " high pressure left indicator.
- 1 " " low pressure left indicator.
- 2 " " dynamometer car records.

The run consisted of a trip from Susquehanna Pa. to Gulf Summit N.Y. This piece of track is about eight miles long and has a heavy grade varying from % to % , with several very heavy curves . The engines are used for pushing East bound freight up this grade, and are at present doing the work formerly performed by 3 and 4 decapod type engines. The run lasts between an hour and an hour and a half depending on condition of track, load, etc..

Description of the Locomotive.

The boiler of this locomotive is designed for 215# per sq. in.. It is of the radial stay type, with a length overall of 43 ft. It is equipped with a combustion chamber of 4 ft. in depth, radially stayed to the shell of the boiler. The widest part of the boiler is at the back end base, measuring 126 inches.

The back head has a slope from base to top of 16" in a distance of $99\frac{3}{4}$ ", the height of the boiler at this end. The distance between the back head and the fire-box sheet is 6". The fire-doors are two in number, and are 20" x 16".

The fire box is of the Wooten type and is 126" long x 114" wide, with a depth at the back end of 72", being provided with a rocking grate. The water space at the mud-ring is 5" on all sides.

The thickness of the boiler plate varies from $1\frac{1}{8}$ " at the gusset to 1" at the sheet next the smoke-box. The thickness of the smoke-box sheet is $\frac{1}{2}$ ". The back head has a thickness of $\frac{5}{8}$ ", while the fire-box sheet is $\frac{3}{8}$ ".

There are 404 tubes of charcoal iron, weighing 23,700#, each having a length of 21 ft. and an outside diameter of $2\frac{1}{4}$ ". The tube sheet at the combustion chamber end is $\frac{9}{16}$ " thick, while at the smoke-box end, the sheet is $\frac{3}{4}$ ". The smoke-box is

about 84" long, and has an outside diameter of 87". The weight of the boiler complete is 97,200#, and it holds 42,700# of water.

The receiver pipe is 9" in diameter, and extends forward from the cylinder saddle, being connected to the latter by a ball-joint. It is made up of three sections to facilitate removal for repairs. At the front end it is connected to a curving Y pipe by means of a slip-joint, allowing variations in length due to curving. Thru this Y pipe steam passes to the low pressure cylinders. The receiver pipe allows a 16 degree curve to be passed over. Steam for the low pressure cylinders exhausts thru a flexible pipe connection to the exhaust pipe in the smoke-box.

The ¹part of the weight of the boiler resting on the forward engine is supported by a self-adjusting sliding bearing located between the third and fourth driving wheels. This bearing consists of a built up saddle casting, which extends down, bearing on a cast steel cross-tie directly below it thru a wrought iron, case hardened sliding plate. Brasses are introduced between the boiler bearing casting and the wrought iron plate. The sliding plate is planed radially on the bottom, adjusting itself to the alignment of the engine; the load on the sliding plate is at all times perfectly distributed, there being no cutting of the wearing surfaces. Movement in a vertical direction is

prevented by a safety connection between the boiler bearing casting and the cross tie, which prevents the frames from hopping away from the boiler in case of derailment. There is also a similar connection provided at the front end of the boiler between the guide yoke casting and the exhaust pipe below.

Another sliding support is located between the second and third pair of driving wheels. This support is so adjusted that it does not take any of the load except under unusual conditions when inequalities in the road bed make it necessary. It is provided with a floating balanced device, which serves to take some of the load off the main boiler bearing. This device consists of a pair of columns, one on either side of the center of the engine, free to sway as the engine turns thru curves; these columns have a ball-joint connection at the upper ends, with a saddle castings bolted to the boiler and a ball-joint connection at the lower ends with flap castings hinged to the bottom of a cross-tie across the lower rails of the frame. Around the outer ends of these hinged castings are U bolts the horns of which extend up thru the bottom of the cross-tie, and thru coiled springs seated on the cross-tie. These springs thus exert an upward force on the columns equal to the total compression of the springs. The initial total compression is 30,000#, which can be increased by

screwing down the spring caps by means of nuts on the ends of the bolts. This boiler support is also provided with a spring centering device. Another sliding support is formed between the exhaust pipe elbow and the guide yoke casting, also forming a connection between the frames and the boiler.

A novel design of throttle valve has been fitted to these locomotives, which in addition to taking steam at the top only, also acts as a steam separator. The arrangement is such that the entering steam strikes against a curved surface on which the entrained water will be deposited, and following this surface will pass down thru the center of the valve to an outlet belw. The top of the casting does not take a bearing and hence does not in any way act as a valve. The steam is led from the throttle pipe thru a short dry-pipe to a joint directly above the high pressure cylinders, where it passes thru the shell to a T-head on top of the boiler, and thence thru wrought iron steam pipes on either side to the top of the high pressure steam chest.

The intercepting valve is simple in operation. In starting live steam enters thru the head and causes the valve to move due to differential areas being exposed to the pressure, causing unbalancing. This movement closes the high pressure exhaust and allows the steam to enter the receiver thru a reducing valve. After a few revolutions acting

sufficient pressure is accumulated by the exhaust of the high pressure cylinder to force it open against the pressure of the live steam on the differential area of the reducing valve. This opens the intercepting valve and at the same time closes the reducing valve. The engine is then working compound. The reducing valve being loose on the stem of the main valve is capable of sufficient movement to close the live steam inlet independently of the movement of the main valve, thus working as an ordinary differential reducing valve to restrict the pressure in the low pressure cylinder to the amount determined upon in equalizing the work of the two cylinders.

In case it is desired to work the engine simple for a longer period in starting the emergency exhaust valve is opened admitting steam behind the piston by means of a three way cock in the engineer's cab, which allows the exhaust from the high pressure cylinder to escape to the atmosphere and prevents the accumulation of pressure behind the main valve. In changing from compound to simple the balancing piston and chamber, features peculiar to the Mellin compound, come into play. The balancing chamber is of small volume, opening into the receiver thru six $\frac{3}{4}$ " holes in the balancing piston, which ordinarily suffice to equalize the pressure between them. But upon opening the emergency exhaust valve, this chamber is emptied more rapidly than

the area of these holes can supply it, with the result that the intercepting valve is closed instantly by the pressure on the balancing piston. This prevents the loss of pressure in the receiver which would be necessary to shut the valve in the ordinary manner, and avoids a drop in the power of the engine right at the sticking point.

A by-pass valve is used on the low pressure cylinders whose office is to prevent an air pump action of the pistons while drifting; this action would create a draught thru the exhaust pipe as good as that when working under steam. When working under steam, the pressure from the steam chest keeps these valves closed. But on shutting the throttle a partial vacuum is soon formed in the steam-chest, which acting on the ends of the valves causes them to open and establish direct communication between the opposite inds of the cylinders. Owing to the extreme width of the fire-box it was necessary to place the cab over the boiler shell near the front, and hence all the controlling apparatus, injectors, etc., are located on the right hand side.

Inspite of the fact that the locomotive weighs 409,900#, it has a weight per driving axle that is less than many large consolidation freight locomotives, thus allowing an enormous amount of power to be centered in one machine, which is still capable of operating over the same track that other

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heavy freight locomotives use. The frames are of cast steel; special attention being paid to obtain a thorough system of cross-bracing. The front and rear frames are joined by a 5" pin connection thru castings secured ahead of the high pressure cylinder. The 2" vertical bolts connecting the upper rail of the front frame with the lower rail of the rear frame are fitted with ball-joints to permit free movement of the two frames relative to each other, and are provided to hold the frames in line and prevent binding on the hinged connection.

Of the 16 journal boxes, the four under the main drivers, the third in each group, are 10"x13", while those under the other drivers are 9"x13". The drivers are 51" in diameter.

The high pressure cylinders are cast in pairs with saddles, the separation between the two cylinders being $8\frac{1}{2}$ " to the right of the center. This permits the intercepting valved to be placed in the left cylinder casting, and also gives room for the connection to the receiver pipe. The low pressure cylinders are cast in pairs, the connection to the receiver pipe being made thru a Y shaped casting connected at the ~~base~~ to the cored passages in the cylinders. The exhaust is carried thru an elbow located at the top, and in the center, to a short pipe with universal joints leading to the exhaust pipe in the front

end. The high pressure cylinders are fitted with piston valves having internal admission, and the low pressure cylinders have balanced Richardson slide valves with external admission. The diameter of the high pressure cylinders is 25", of the low pressure is 39". This is in accordance with the general principle in the Mallet compounds of having the ratio about $1\frac{1}{2}$ to 1, thus equalizing the work done by the high and low pressure cylinders. The stroke is 28".

The valve gear, which is of the Walschaert type, is so arranged that the return crank leads the pin in both sets, and hence the block is at the bottom of the link for forward motion for the low pressure cylinders and at the top of the link for the high pressure cylinders. In this way the weight of the two valves counterbalancing.

The operation of reversing is further assisted by a pneumatic reversing device, which is connected to the reverse lever and consists of two cylinders, one of which contains oil under pressure for locking the device in any desired position, the other cylinder being the air cylinder. The operation of this device is controlled from an auxiliary reversing lever in the cab.

General Data.

Guage	4 ft. 8 $\frac{1}{2}$ in.
Service	Pushing
Fuel	2 Bituminous Coal
Tractive effort, compound	94,800 $\frac{1}{2}$
Weight in working order	409,000 $\frac{1}{2}$
Weight on drivers	409,000 $\frac{1}{2}$
Wt. engine and tender, working order	572,000 $\frac{1}{2}$
Wheel base, driving	39ft. 2 in.
Wheel base, total	39ft. 2 in.
Wheel base, rigid	14ft. 3 in.
Wheel base, total, engine and tender	70ft. 5 $\frac{1}{2}$ in.
Brake	New York Air

Ratios.

Weight on drivers \div tractive effort	4.32
Total weight \div tractive effort	4.32
Tract.ef. \times diam.drivers \div heat,sf.	910.00
Total heat.surf. \div grate area	53.14
Fire box sf. \div total heat. sf. $\%$	6.46
Weight on drivers \div total heat. sf.	76.90
Volume both cylinders, cu.ft.	24.00
Total heat sf. \div volume cylinders	222.00
Grate area \div volume cylinders	4.17

Cylinders.

Kind	Mellin compound
Number	4
Diameter	25" and 39"
Stroke	28"
Piston rod, diameter	4 $\frac{1}{2}$ "

Cylinder Clearances.

Vol of clearance, H.E. & C.E. H.P. cyl,	10.2%
" " " " " " " " " " " " " "	1400 cu.in
" " " " " " " " " " " L.P. " "	8.3%
" " " " " " " " " " " " " "	2500cu.in
" " receiver cylinder, pump & pipe	37 cu.ft.

Wheels .

Driving, diameter over tires	51"
Driving, thickness of tires	3 $\frac{1}{2}$ "
Driving journals, main, diameter and length	10"x13"
Driving journals, others, " " " " " " " "	9"x13"

Boiler

Type	Radial stay
Style	Straight with Conical Connection
Working pressure	215#
Outside diameter of first ring	84"
Firebox type	Wooten
Firebox, length and width	126 $\frac{1}{8}$ "x114 $\frac{1}{4}$ "
Firebox plates, thickness	3/8" and $\frac{1}{2}$ "
Firebox water space	5"
Tubes, material	Charcoal iron
Tubes, number	404
Tubes, guage	0.125mm.
Tubes, outside diameter	2 $\frac{1}{2}$ "
Tubes, length	21'
Heating surface, tubes	4,971.5 sq.ft.
Heating surface, firebox	342.2 sq.ft.
Heating surface, total	5313.7 sq. ft.
Stack diameter	18"

Stack, height above rail	15ft. 5 $\frac{3}{4}$ "
Center of boiler above rail	120"

Tender.

Tank	Water bottom
Frame	12"-40# Channels
Water Capacity	8500 gals.
Coal Capacity	16 tons
Wheels diameter	33"
Journals, diameter and length	5 $\frac{1}{2}$ " x 10"

Conclusions

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Below is given a table of results as obtained on the Pennsylvania R.R. tests, at the St. Louis Exposition on freight locomotives running at about the same speeds as the locomotive under test:-

	P.R.R.	Erie.
Speed, mi. per hr	6.7-7.5	5.0-6.5
I.H.P.	454-633	800-1141
D.H.P.	373-557	584-999
Dynamometer push, lbs.	20800-31100	41000-66000
Mech. Eff.	82.2-94	73.0-87.1
Dry coal per sq' grate area per hour	21.7-60.3	34.2-49.1
Dry steam per sq' heating surface per hour	3.5-6.1	3.3-7.9
Dry steam per lb. dry coal	7.6-10.1	5.1-10.0
Lbs. steam per I.H.P.	20.3-27.3	22.0-37.5
" " " " " D.H.P.	21.6-33.2	30.0-46.3
Draft in inches of water	0.9-3.1	3.0-5.0
Efficiency of boiler	58.6-78.4	48.0-80.5
Heat efficiency of locomotive	42-8.0	2.0-5.1

These comparisons tend to show that the locomotive compares favorably with modern standard freight practice in locomotives at about the same speed.

The test of this locomotive was made in actual service, no special repairs having been made on the locomotive previous to the test. Difficulty was experienced in the third and fourth runs with the blower, causing the engine to steam poorly.

The fact that these engines have taken the place of three pushers, allows savings to be made in fuel, supplies, crew, repairs, and round-house labor.

During the six months in their service, the tires by actual measurment had worn from 51 inches to $50\frac{5}{8}$ inches, the wear being uniform on all the driv- (ers)

Considering the conditions under which this locomotive has been during the time it has been in service, i.e., severe grade, wintry weather, and severe and sudden variations in load, it is in very good condition, not having been laid up for serious repairs as yet.

Of these three locomotives, one is kept in reserve, the other two pushing an average of seven or eight trains each day of 24 hours up the grade from Susquehanna to Gulf Summit. These trains average about 60 to 70 loaded cars, or about 3000 to 3500 tons.

As to the dynamometer records, these show the steady pushing obtainable from this type of engine. In various parts of the records, severe vibration appeared, indicating slip as was marked on one of the plates shown. Quite frequently, however, straight lines were recorded for some distance. An example of this also appearing on the print. This line indicates the steady even push that was given when the engine was working compound, steam being admitted, and cut off occurring at the same time in

the cylinders on the same side of the engine. There were some places in the record where the push was greater than 70,000#, the capacity of the car. In such a case, the recording lever was unfastened, these cases, however were few and ~~far~~ in all lasted for a very short period.

The coal used on the test was Bituminous and was from the Dagus Mine of the Erie R.R. A sample of coal was taken from each run, pulverized, and placed in a glass jar, analyses were taken of these separate samples, but did not prove satisfactory, so they were mixed and an average analysis taken, which was used in working up the data.

Moisture in the coal was about 3.1%, the ash about 15.7%, ~~11%~~ volatile matter 28.4% leaving fixed carbon of 52.8 %. The calorific value of the coal was 12,000 B.T.U. per pound.

Analysis of the flue gas pointed out the good service rendered by the combustion chamber, the percentage of CO being very small.

CALIBRATION.

Thermometers

Standard	#6	#4	#1	#3
201	205	238	204	228
219	222	256	222	244
239	242	275	242	264
262	263	296	262	286
268	262	297	262	288
281	282	316	280	305
298	298	332	297	322
319	318	352	317	341
338	337	370	335	360
360	356	390	356	380
385	380	414	380	404
402	396	431	396	421

Standard	#5	Standard	#2
35	38	40	38
48	60	50	48
55	63	62	61
65	69	70	70
75	75	80	80
85	83	90	91
95	92	100	102
105	106	110	106

CALIBRATION.

Indicator Springs.

Actual Pressure #/sq. in.	R.L.P. Spring #/sq.in.	L.L.P. Spring #/sq. in.	R.H.P. Spring #/sq. in.	L.H.P. Spring #/sq.in.
10	11.0	11.0	10.3	10.5
20	21.0	21.5	19.5	19.5
30	31.5	31.5	29.5	29.0
40	41.0	41.5	37.5	37.5
50	50.5	51.5	47.5	47.0
60	60.5	62.5	56.5	57.5
70	70.5	71.5	66.0	67.0
80	80.5	82.0	75.0	76.0
90	90.5	92.0	85.0	85.0

CALIBRATION.

Height of water	Tank (1)	
	Gross weight in pounds	Net Weight in pounds.
5-11½	138,000	70,000
5-9½	136,740	68,740
5-8½	135,900	67,900
5-7	134,810	66,810
5-6	134,000	66,000
5-5	133,200	65,200
5-4	132,580	64,580
5-3	131,620	63,620
5-2	131,000	63,000
5-1	130,200	62,200
5-0	129,240	61,240
4-11	128,500	60,500
4-10	127,680	59,680
4-9	126,900	58,900
4-8	126,070	58,070
4-7	125,330	57,330
4-6	124,400	56,400
4-5	123,540	55,540
4-4	122,720	54,720
4-3	121,980	53,980
4-2	120,800	52,800
4-0½	119,700	51,700
3-11	118,200	50,200
3-10	117,360	49,360
3-9	116,500	48,500
3-7½	115,240	47,240

CALIBRATION

Tank (2).

Height of water	Gross weight in pounds	Net weight in pounds
3-6	113,980	45,980
3-5	113,000	45,000
3-4	112,040	44,040
3-3	111,080	43,080
3-2	110,100	42,100
3-1	109,050	41,150
3-0	108,200	40,200
2-11	107,270	39,270
2-10	106,340	38,340
2-9	105,380	37,380
2-8	104,400	36,400
2-7	103,480	35,480
2-6	102,550	34,550
2-5	101,550	33,550
2-4	100,560	32,560
2-3	99,580	31,580
2-2	98,600	30,600
2-1	97,590	29,590
2-0	96,580	28,580
1-11	95,540	27,540
1-10	94,500	26,500
1-9	93,600	25,600
1-8	92,700	24,700
1-7	91,620	23,620
1-6	90,540	22,540
1-5	89,240	21,240

Tank (3).

Height of water	Gross weight in pounds	Net weight in pounds
1-4	87,940	19,940
1-3	86,670	18,670
1-2	85,400	17,400
1-1	83,050	15,050
1-0	82,700	14,700
0-11	81,400	13,400
0-10	80,100	12,100
0-9	78,660	10,660
0-8	77,160	9,160
0-7	75,960	7,960
0-6	74,700	6,700
0-5	73,300	5,300
0-4	71,800	3,800
0-3	70,620	2,620
0-2	69,860	1,860
0-1	69,000	1,000
0-0	68,000	-----

COAL ANALYSIS.

	%
Moisture	3.1
Volatile Matter	28.4
Fixed Carbon	52.8
Ash	15.7
B.T.U. per pound	12,100

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COAL

Tare equals 25#.

Run (1)

No. of reading	Gross weight, #	Net weight, #.
1	303	278
2	317	292
3	335	310
4	318	293
5	315	290
6	300	275
7	315	290
8	322	297
9	315	290
10	310	285
11	293	268
12	300	275
13	330	305
14	322	297
15	332	307
16	325	300
17	299	274
18	291	266
	Subtract	<u>5192.0</u>
	152	<u>127.0</u>
	Weight burned	<u>5065.0</u> pounds.

COAL

46

Tare equals 25 $\frac{1}{2}$ lbs.

Run (2)

No. of reading	Gross weight, #	Net weight, #
1	308	283
2	332	307
3	322	297
4	323	298
5	319	294
6	323	298
7	311	286
8	321	296
9	332	307
10	331	306
11	319	294
12	344	319
13	327	302
	Subtract	<u>3887</u>
	201	176
	Weight burned	<u>3711</u> pounds.

COAL

47

Tare equals 25 $\frac{1}{2}$ lbs.

Run (3)

No. of reading	Gross weight, #	Net weight, #
1	322	297
2	325	300
3	320	295
4	325	300
5	334	309
6	335	310
7	334	309
8	325	300
9	336	311
10	332	307
11	334	309
12	338	313
13	333	308
14	322	297
15	323	298
16	340	315
17	331	306
18	318	293
19	317	292
20	311	286
21	303	278
22	315	290
23	317	292
24	328	303
Weight burned		<u>7218.</u> pounds.

COAL

48

Tare equals 25#.

Run (4)

No. of reading	Gross weight, #	Net weight, #.
1	327	302
2	335	310
3	328	303
4	325	300
5	309	284
6	307	282
7	308	283
8	300	275
9	304	279
10	263	238
11	280	255
12	325	300
13	310	285
14	315	290
	Weight burned	<u>3986.</u> pounds.

WATER

Run (1)

Beginning of run	5'-0 ⁷ / ₈ "	130,080 #
End of run	1'-3"	<u>86,670 #</u>
Total weight of water		43,410 #

Run (2)

Beginning of run	5'-3 ³ / ₄ "	132,340 #
End of run	3'-3 ³ / ₄ "	<u>111,800 #</u>
Total weight of water		20,540 #

Run (2)

Beginning of run	5'-3"	131,620 #
End of run	1'-6 ³ / ₄ "	<u>91,350 #</u>
Total weight of water		40,270 #

Run (3)

Beginning of run	5'-3"	131,620 #
End of run	1'-0"	<u>82,700 #</u>
Total weight of water		48,920 #

PYROMETER, DRAFT, EXHAUST TEMPERATURE.

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Run (2) /

No. of reading	Pyrometer degrees F.	Draft inches of water	Exhaust temperature degrees F.
1	510	4.4	207
2	510	4.6	202
3	510	5.0	207
4	510	5.0	202
5	510	4.0	202
6	510	4.0	202
7	510	5.0	202
8	510	4.5	202
9	510	5.5	202
10	510	5.5	202
11	510	5.5	202
12	510	4.0	202
13	510	5.0	202
14	510	5.5	202
15	510	5.5	202
16	510	5.5	202
17	510	5.0	202
18	510	5.5	202
19	510	5.5	202
20	510	3.5	202
21	510	4.0	202
22	510	4.0	202

PYROMETER, DRAFT, EXHAUST TEMPERATURE.

No. of reading	Run ($\frac{2}{12}$)		
	Pyrometer degrees F.	Draft inches water	Exhaust temperature degrees F.
1	510	4.5	194
2	510	4.0	194
3	510	4.0	194
4	510	4.0	194
5	510	4.0	196
6	510	5.0	196
7	510	5.5	198
8	510	6.0	202
9	510	5.5	200
10	510	5.5	200
11	510	4.5	201
12	510	4.5	200
13	510	5.5	202
14	510	5.5	196
15	510	5.5	198
16	510	5.0	198
17	510	4.5	198
18	510	4.0	198
19	510	3.0	200
20	510	5.0	192
21	510	5.0	191
22	510	4.0	191
23	510	4.0	191

PYROMETER, DRAFT, EXHAUST TEMPERATURE.

Run (3)

52

No. of reading	Pyrometer degrees F	Draft, inches of water	Exhaust temp. degrees F.
1	510	4.0	222
2	510	5.0	222
3	510	4.0	224
4	510	4.0	230
5	510	2.0	235
6	510	0.0	240
7	510	4.0	230
8	510	4.0	240
9	510	4.5	235
10	510	5.0	238
11	510	4.5	238
12	510	4.5	240
13	510	4.0	238
14	510	3.0	240
15	510	6.0	240
16	510	4.0	244
17	510	4.0	250
18	510	3.0	254
19	510	4.0	244
20	510	4.5	252
21	510	4.5	240
22	510	4.0	230
23	510	5.0	238
24	510	5.5	240
25	510	5.0	236
26	510	3.0	242
27	510	6.0	254
28	510	4.5	256

PYROMETER, DRAFT, EXHAUST TEMPERATURE.

53

Run (4)

No. of reading	Pyrometer degrees F	Draft, inches of water	Exhaust temp degrees F.
1	500	4.0	215
2	470	4.0	224
3	450	3.5	222
4	440	2.5	230
5	430	1.0	230
6	430	2.0	222
7	490	3.0	218
8	510	4.5	218
9	490	4.5	222
10	470	4.0	220
11	450	2.5	220
12	410	2.5	222
13	400	1.5	222

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High Pressure Calorimeter

Run (1)

No. of reading	Pressure in pounds	Temperature in degrees F	Manometer in. of Hg	Quality %
1	---	---	---	---
2	---	---	---	---
3	---	---	---	---
4	175	270 241/25/	14.25 270	99.2
5	190	273	12	98.4
6	180	270	15	99.0
7	180	268	11/25	99.1
8	185	268	10.5	99.0
9	190	270	12	99.3
10	180	271	15.25	99.2
11	180	270	14	99.2
12	190	265	14	98.5
13	194	264	14.25	98.0
14	195	272	15	98.9
15	195	276	15.25	99.4
16	185	276	15.	99.5
17	180	272	15	99.1
18	190	273	14.75	99.3
19	185	273	15	99.3
20	182	272	15	99.3
21	180	272	14.75	99.3
22	180	272	14.5	99.3
23	180	272	14.75	99.3

Average quality 99.08 %

Run 12)

No. of reading	Pressure in pounds	Temperature in degrees F	Manometer in. of Hg	Quality %
1	175	282	15	97.7
2	175	273	15.5	97.6
3	180	278	15	97.7
4	175	280	15.5	97.6
5	180	278	17	97.4
6	185	278	17	97.1
7	180	285	16.5	97.9
8	180	282	15	97.5
9	170	284	15	98.0
10	170	284	14	97.9
11	165	280	16.5	97.8
12	175	278	17	97.4
13	185	282	15	97.7
14	175	285	16	97.6
15	165	283	14	97.7
16	175	278	16	97.2
17	160	278	14	97.8
18	155	276	14	97.7
19	160	276	14	97.8
20	160	274	14	97.6
21	160	276	14.	97.7
22	160	288	14	97.4

Average quality 97.65 %

High Pressure Calorimeter

Run (3).

No. of reading	Pressure in pounds	Temperature in degrees F.	Manometer in. of Hg	Quality %
1	175	298	13.5	98.7
2	175	298	13.5	98.8
3	170	296	13.5	97.3
4	168	296	13.	97.5
5	163	296	13.	98.7
6	160	295	22.5	98.5
7	190	286	16.5	97.5
8	190	296	17.	98.1
9	180	298	16.5	98.5
10	175	298	16	98.5
11	170	298	15	98.7
12	173	298	15	98.5
13	160	298	14.25	97.8
14	180	298	16.5	98.1
15	186	296	16.5	98.8
16	185	298	18	98.9
17	180	298	16.5	98.3
18	177	296	19.5	98.4
19	175	296	16	98.5
20	165	296	14.5	98.4
21	172	298	15.25	98.6
22	185	298	17.5	98.6
23	190	298	19	98.6
24	173	298	16	98.6
25	185	300	18.25	98.6
26	195	300	19	98.6
27	184	300	18.5	98.6
28	160	298	15.5	98.6
Average quality				98.39 %

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High Pressure Calorimeter.

57

Run (4).

No. of reading	Pressure in pounds	Temperature in degrees F.	Manometer in. of Hg	Quality %
1	145	294	10	99.0
2	135	293	10	99.0
3	125	288	7.5	99.0
4	120	280	7.5	98.7
5	112	276	6.5	98.8
6	163	278	13	97.8
7	157	284	12.25	98.2
8	155	288	12.3	98.4
9	135	292	9.5	99.0
10	125	288	9.	99.0
11	120	286	9	98.9
12	112	280	7.5	98.8
13	107	280	6.5	98.9

Average quality 98.73 %

Low Pressure Calorimeter.

5.8

Run (2).

No. of reading	Pressure in pounds	Temperature degrees F	Quality %
1	50	228	97.9
2	50	228	97.9
3	50	228	97.9
4	50	228	97.9
5	50	228	97.9
6	50	228	97.9
7	53	230	98.0
8	53	228	98.4
9	50	236	98.4
10	50	236	98.4
11	50	230	98.1
12	50	230	98.1
13	50	232	98.2
14	50	232	98.2
15	50	232	98.2
16	50	230	98.1
17	50	230	98.1
18	48	230	98.1
19	47	230	98.2
20	48 -	224	97.8
21	48	222	97.7
22	43	222	97.9
23	43	222	97.9

Average quality 98.05 %

Low Pressure Calorimeter

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No. of reading	Pressure in pounds	Run (3)		Manometer inches of Hg	Quality %
		Temperature in degrees F.			
1	50	238		0.75	98.5
2	50	238		0.75	98.5
3	50	236		0.75	98.4
4	50	235		0.75	98.3
5	50	230		0.75	98.1
6	50	230		0.75	98.1
7	60	234		1.00	97.9
8	60	236		1.00	98.1
9	60	238		1.00	98.2
10	60	238		1.00	98.2
11	55	238		0.80	98.3
12	55	238		0.80	98.3
13	55	238		1.00	98.2
14	55	232		0.80	98.0
15	60	232		1.05	97.8
16	60	233		1.25	97.8
17	60	230		1.00	97.7
18	50	232		1.25	97.8
19	55	230		1.00	97.8
20	55	236		0.80	98.2
21	60	238		1.25	98.2
22	60	238		1.00	98.2
23	60	240		1.25	98.3
24	55	240		1.00	98.4
25	60	240		1.25	98.3
26	60	236		1.25	98.1
27	60	230		1.25	97.7
28	50	236		1.25	98.2

Average quality 98.14%

LOW Pressure Calorimeter.

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Run (4).

No. of reading	Pressure in pounds.	Temperature in degrees F	Manometer inches of Hg.	Quality %
1	50	260	2.00	99.6
2	48	262	2.00	99.7
3	45	258	2.25	99.6
4	40	254	2.50	99.6
5	40	254	2.50	99.6
6	50	252	2.50	99.4
7	55	256	2.25	99.2
8	55	262	2.50	99.5
9	50	260	2.50	99.6
10	45	258	2.25	99.6
11	40	256	2.50	99.7
12	40	254	1.00	99.6
13	38	252	1.00	99.7
Average quality				99.56%

Card Data.

Run 1. Right Low Pressure.Cylinder.

Head End.

No. of card.	Area, in sq. in.	Length, in in.	M.E.P. in #	I.H.P.	% cut off.
1	1.66	3.00	33.8	123	75.0
2	---	---	---	---	---
3	---	---	---	---	---
4	---	---	---	---	---
5	2.37	3.65	38.7	127	73.2
6	1.76	3.27	32.0	100	72.5
7	1.56	2.78	33.3	115	60.2
8	2.19	3.25	40.0	139	70.6
9	1.75	3.27	31.8	116	67.0
10	1.85	3.33	33.0	125	78.8
11	2.48	3.30	44.7	144	73.9
12	2.09	3.27	38.0	122	71.1
13	1.73	3.25	31.7	110	71.2
14	2.03	3.29	36.6	133	73.0
15	2.01	3.32	36.0	140	69.5
16	1.72	3.23	31.7	115	30.4
17	1.61	2.69	35.4	129	72.5
18	2.17	3.27	38.8	125	71.4
19	2.20	3.32	39.4	100	71.5
20	2.23	3.46	38.4	117	73.4
21	2.51	3.48	43.0	189	77.3
22	2.13	3.50	36.0	---	70.9

Card Data.

Run 1. Right Low Pressure Cylinder.

Crank End.

No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	1.76	3.00	34.6	125	67.6
2	---	---	---	---	---
3	---	---	---	---	---
4	---	---	---	---	---
5	2.33	3.65	38.0	124	72.7
6	2.06	3.27	37.4	118	73.5
7	1.62	2.71	34.3	117	72.0
8	2.07	3.25	37.8	138	70.6
9	1.49	3.27	28.8	103	72.8
10	1.82	3.33	32.4	122	75.4
11	2.20	3.30	39.7	125	77.5
12	2.24	3.27	40.9	139	65.0
13	1.85	3.25	33.8	116	72.7
14	1.84	3.29	33.2	122	72.0
15	1.87	3.32	33.4	128	60.3
16	1.80	3.23	33.2	119	68.5
17	1.56	2.69	34.5	124	71.7
18	1.84	3.27	34.8	110	67.3
19	2.02	3.32	36.2	91	75.0
20	2.05	3.46	35.3	97	70.8
21	2.23	3.48	38.2	170	66.4
22	2.26	3.50	38.4	---	68.4

Card Data.

Run 1.

Left Low Pressure Cylinder.

Head End.

No. of card.	Area, in sq. inches.	Length in, in.	M.E.P. in $\frac{H}{H}$	I.H.P.	% cut off
1	2.35	3.83	36.0	131	73.1
2	2.41	3.79	37.4	129	65.0
3	2.40	3.74	37.8	131	65.0
4	2.61	3.78	40.5	140	68.0
5	2.53	3.76	39.7	131	68.4
6	2.74	3.81	42.0	131	61.0
7	2.44	3.77	37.8	131	69.5
8	2.51	3.80	38.5	143	79.5
9	2.44	3.79	37.8	137	72.8
10	2.69	3.80	41.4	157	72.5
11	2.42	3.77	37.5	117	68.3
12	2.45	3.79	37.6	118	73.9
13	2.68	3.78	41.4	143	75.8
14	2.56	3.82	39.6	144	76.5
15	2.43	3.77	37.8	147	68.2
16	2.32	3.73	36.3	132	67.6
17	2.36	3.75	36.2	131	74.2
18	2.55	3.80	39.7	127	74.3
19	2.29	3.78	35.6	91	73.8
20	2.69	3.75	42.0	117	73.5
21	2.44	3.78	37.8	167	76.7

Card Data.

Run 1. Left Low Pressure Cylinder.

Crank End.

No. of card.	Area , in sq. in.	Length, in in.	M.E.P. in #	I.H.P.	% cut off.
1	2.43-	3.83	3.73	133	70.0
2	2.54	3.79	39.4	134	67.3
3	2.52	3.74	39.6	136	66.8
4	2.48	3.78	38.6	132	72.7
5	2.72	3.76	42.5	138	77.4
6	2.64	3.81	40.8	126	72.5
7	2.45	3.77	38.2	131	75.6
8	2.55	3.80	39.4	144	77.5
9	2.65	3.79	41.2	148	73.1
10	2.69	3.80	41.8	157	69.3
11	2.45	3.77	38.2	121	72.1
12	2.59	3.79	40.2	127	72.5
13	2.73	3.78	42.7	146	78.0
14	2.59	3.82	39.9	146	71.3
15	2.50	3.77	39.0	149	73.2
16	2.28	3.73	35.9	129	66.7
17	2.34	3.75	36.7	132	72.0
18	2.42	3.80	37.6	119	72.6
19	2.50	3.78	39.0	97	72.8
20	2.61	3.75	40.0	110	68.3
21	2.35	3.78	36.5	158	75.5

Card Data.

Run 1. Left High Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% out off.
1	2.47	3.77	105.5	165	65.1
2	2.37	3.63	103.9	156	59.0
3	2.32	3.39	107.4	145	78.2
4	2.54	3.62	109.6	153	69.9
5	2.69	3.65	105.7	135	72.1
6	2.77	3.60	120.6	146	72.5
7	2.62	3.67	111.6	143	72.2
8	2.65	3.65	113.9	154	74.3
9	2.65	3.61	114.9	163	74.8
10	---	---	---	---	---
11	---	---	---	---	---
12	---	---	---	---	---
13	---	---	---	---	---
14	---	---	---	---	---
151	---	---	---	---	---
16	---	---	---	---	---
17	---	---	---	---	---
18	---	---	---	---	---
19	---	---	---	---	---
20	---	---	---	---	---
21	---	---	---	---	---
22	---	---	---	---	---

Card Data.

Run 1. Left High Pressure Cylinder.

Crank End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in ²	I.H.P.	% out off-
1	2.68	3.77	105.0	159	74.0
2	2.49	3.63	107.9	166	70.4
3	2.21	3.39	87.8	115	76.7
4	2.63	3.62	114.3	153	73.2
5	2.55	3.65	110.0	137	73.0
6	2.82	3.60	123.4	145	72.2
7	2.77	3.67	119.9	148	72.7
8	2.61	3.65	112.3	147	76.2
9	2.51	3.61	109.3	151	75.3
10	---	---	---	---	---
11	---	---	---	---	---
12	---	---	---	---	---
13	---	---	---	---	---
14	---	---	---	---	---
15	---	---	---	---	---
16	---	---	---	---	---
17	---	---	---	---	---
18	---	---	---	---	---
19	---	---	---	---	---
20	---	---	---	---	---
21	---	---	---	---	---
22	---	---	---	---	---

Card Data.

67

Run 1, Right High Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in $\frac{1}{2}$	I.H.P.	% out off.
1	2.85	3.42	130.5	204	72.8
2	2.52	3.40	116.5	170	70.9
3	2.36	3.41	107.3	145	72.5
4	2.67	3.40	123.0	171	75.0
5	2.58	3.45	117.0	150	67.6
6	2.63	3.41	120.7	146	76.3
7	2.74	3.43	125.0	160	70.9
8	2.54	3.39	117.8	158	75.0
9	2.60	3.40	120.0	170	77.6
10	2.42	3.42	110.4	154	73.6
11	2.55	3.45	119.4	168	78.8
12	2.51	3.37	117.3	172	75.7
13	2.63	3.37	122.3	179	75.4
14	2.66	3.43	121.4	186	74.0
15	2.65	3.37	123.3	201	72.7
16	2.50	3.44	113.9	166	75.0
17	2.55	3.45	125.7	184	74.4
18	2.79	3.47	126.2	151	74.3
19	2.52	3.47	114.7	119	74.8
20	2.75	3.35	127.4	138	76.6
21	2.58	3.45	118.6	193	76.8

Card Data.

Run 1. Right High Pressure Cylinder.

Crank End.

No. of card	Area in sq.in.	Length in in.	M.E.P.E in #	I.H.P.	% cut off.
1	2.61	3.42	115.0	174	67.5
2	2.78	3.40	127.9	180	64.2
3	2.14	3.41	100.0	131	63.8
4	2.36	3.40	108.3	145	69.1
5	2.41	3.45	109.2	136	69.3
6	2.55	3.41	117.2	137	69.5
7	2.49	3.43	113.9	141	68.0
8	2.32	3.39	107.0	148	69.0
9	2.29	3.40	105.5	138	69.4
10	2.21	3.42	102.0	144	70.4
11	2.37	3.45	112.3	138	63.4
12	2.32	3.37	107.8	149	67.1
13	2.72	3.37	126.9	179	66.8
14	2.38	3.43	108.6	161	59.0
15	2.46	3.37	114.6	181	65.4
16	2.64	3.44	120.1	174	59.6
17	2.48	3.44	112.8	163	59.0
18	2.26	3.47	103.0	128	63.4
19	2.33	3.47	101.2	103	63.5
20	2.41	3.35	112.3	117	67.2
21	2.35	3.45	106.2	168	64.0

Card Data.

Run 2, Low Pressure Left Cylinder.

Head End.

No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	2.47	3.77	36.9	133	68.1
2	2.38	3.77	37.4	133	74.2
3	2.26	3.76	35.5	123	83.7
4	2.31	3.72	38.1	127	72.5
5	2.69	3.71	43.1	123	72.2
6	2.57	3.77	40.4	118	74.3
7	2.49	3.73	39.7	132	78.4
8	2.24	3.74	35.5	144	78.3
9	2.41	3.69	38.7	160	77.7
10	2.41	3.76	38.0	158	76.4
11	2.78	3.75	44.0	167	75.8
12	2.65	3.77	41.8	142	77.2
13	2.67	3.80	41.7	126	75.0
14	2.43	3.78	36.0	154	78.0
15	2.34	3.74	37.2	160	78.1
16	2.48	3.73	39.2	153	75.8
17	2.29	3.73	36.5	120	77.0
18	2.27	3.71	36.2	125	77.0
19	2.24	3.71	35.8	118	73.5
20	2.46	3.73	39.2	89	77.0
21	2.54	3.79	40.7	57	83.7
22	1.96	3.74	31.0	46	82.5

Card Data.

Run 2. Left Low Pressure Cylinder.

Crank End.

No. of card.	Area in sq. in .	Length in in.	M.E.P. in $\frac{1}{2}$	I.H.P	% cut off
1	2.31	3.77	35.5	128	63.1
2	2.32	3.77	36.4	127	73.0
3	2.47	3.76	38.9	133	87.6
4	2.61	3.72	41.6	135	66.6
5	2.44	3.71	39.0	110	74.4
6	2.73	3.77	43.0	124	75.0
7	2.52	3.73	40.0	132	76.2
8	2.40	3.75	38.0	152	75.5
9	2.38	3.69	38.2	160	76.8
10	2.40	3.76	37.8	154	77.3
11	2.24	3.75	35.4	132	72.8
12	2.61	3.77	41.0	133	77.7
13	2.63	3.80	41.0	123	74.3
14	2.41	3.78	37.7	151	77.2
15	2.12	3.74	33.5	142	76.8
16	2.45	3.76	38.6	148	75.5
17	2.59	3.73	41.2	134	78.0
18	2.33	3.71	37.2	127	78.8
19	2.33	3.71	37.3	121 222	77.5
20	2.59	3.73	41.2	92	80.3
21	2.52	3.71	40.3	56	82.7
22	2.14	3.74	34.0	50	80.7

Card Data.

Run 2. Right Low Pressure Cylinder.

Head End .

No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	2.29	3.60	37.7	136	71.2
2	2.21	3.60	36.3	129	72.0
3	2.40	3.58	39.8	142	75.2
4	2.44	3.57	40.5	134	74.3
5	2.27	3.52	38.2	111	78.7
6	2.64	3.57	44.0	130	81.2
7	2.46	3.57	40.8	133	79.3
8	2.30	3.59	38.0	154	77.2
9	2.28	3.59	37.7	159	78.0
10	2.30	3.59	38.0	158	78.0
11	2.26	3.59	37.2	141	79.5
12	2.24	3.58	37.2	126	78.5
13	2.50	3.57	41.6	126	79.4
14	2.44	3.59	40.3	154	72.1
15	2.14	3.57	35.5	153	78.5
16	2.28	3.57	37.8	147	77.5
17	2.26	3.57	37.5	123	78.7
18	2.16	3.57	35.8	124	79.0
19	2.23	3.55	37.3	122	72.5
20	2.34	3.56	39.0	88.	71.9
21	2.49	2.57	41.3	58	81.0
22	2.08	3.58	34.5	51	78.4

Run 2. Right Low Pressure Cylinder.

No. of card.	Area in sq."	Crank End.		M.E.P. #	I.H.P.	% cut off
		Length, "	Length, "			
1	2.33	3.60		38.3	137	67.8
2	2.26	3.60		37.2	130	70.4
3	2.24	3.58		37.2	127	68.7
4	2.73	3/57		45.2	146	67.2
5	2.39	3.52		41.2	116	68.3
6	2.52	3.57		42.0	121	76.5
7	2.54	3.57		42.2	133	76.5
88	2.23	3.59		38.8	147	76.0
9	2.12	3.59		35.0	146	73.4
10	2.18	3.59		36.0	147	75.5
11	2.14	3.59		35.3	132	76.6
12	2.29	3.58		38.0	126	73.5
13	2.49	3.57		41.5	124	77.6
14	2.35	3.59		38.7	158	73.0
15	1.99	3.57		33.0	141	71.7
16	2.31	3.57		38.3	146	74.2
17	2.11	3.57		38.3	114	73.7
18	2.12	3.57		35.2	120	74.7
19	2.18	3.55		36.5	118	73.6
20	2.28	3.56		38.3	85	72.2
21	2.15	3.57		35.8	49	78.1
22	1.95	3.58		32.2	37	32.2

Card Data.

Run 2, Left High Pressure Cylinder.

Head End.

No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off
1	2.01	3.40	83.0	118	63.5
2	2.25	3.46	103.2	140	66.5 64.2
3	2.25	3.57	102.0	130	64.5
4	2.30	3.55	102.9	129	65.2
5	2.45	3.57	106.9	119	64.7
6	2.54	3.57	110.5	132	69.9
7	2.48	3.62	108.0	129	73.8
8	2.33	3.67	101.0	169	70.8
9	2.36	3.67	102.2	164	70.0
10	2.28	3.63	99.8	160	71.7
11	2.30	3.63	100.8	147	72.5
12	2.41	3.59	106.0	141	72.7
13	2.67	3.61	116.0	141	70.2
14	2.31	3.64	101.0	137	69.6
15	2.28	3.65	99.3	154	71.8
16	2.48	3.62	108.0	163	73.2
17	2.30	3.60	101.4	127	72.6
18	2.15	3.63	94.1	127	72.6
19	2.22	3.60	98.0	122	70.5
20	2.38	3.53	106.5	88	75.2
21	2.34	3.55	104.0	51	74.0
22	1.96	3.62	86.0	48	68.3

Card Data.

Run 2. Left High Pressure Cylinder.

Crank End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	2.31	3.40	107.2	147	67.5
2	2.22	3.4 6 ¹	102.0	135	66.5
3	2.43	3.57	109.5	137	67.8
4	2.43	3.55	107.8	131	67.0
5	2.40	3.57	106.2	115	66.2
6	2.59	3.61	113.0	132	73.4
7	2.68	3.62	115.3	132	73.0
8	2.44	3.67	105.0	150	71.6
9	2.47	3.67	106.3	164	72.3
10	2.30	3.63	100.3	150	73.6
11	2.35	3.63	102.6	150	72.5
12	2.51	3.59	110.3	141	74.4
13	2.45	3.61	107.0	126	68.5
14	2.33	3.64	101.4	134	72.8
15	2.34	3.65	101.8	154	73.4
16	2.50	3.62	109.0	158	73.7
17	2.44	3.60	107.0	130	74.4
18	2.15	3.63	94.3	124	73.2
19 ¹	2.31	3.60	101.8	124	73.7
20	2.39	3.53	101.6	87	77.0
21	2.34	3.55	103.2	48	76.1
22	2.19	3.62	96.1	52	75.8

Card Data.

Run 2 . Right High Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off
1	2.31	3.41	106.0	151	66.5
2	2.37	3.50	106.3	144	67.5
3	2.39	3.48	107.5	138	68.1
4	2.35	3.44	107.0	135	71.6
5	2.55	3.44	116.3	130	69.8
6	2.70	3.45	122.7	146	74.0
7	2.66	3.51	119.0	140	75.5
8	2.51	3.52	112.0	166	73.4
9	2.45	3.48	110.8	175	78.1
10	2.44	3.46	111.0	175	76.5
11	2.35	3.50	106.0	166	75.7
12	2.45	3.46	111.7	147	77.4
13	2.70	3.48	121.7	149	76.7
14	2.53	3.46	115.0	155	78.0
15	2.42	3.47	110.0	172	75.9
16	2.62	3.46	118.8	178	77.5
17	2.39	3.50	107.8	135	76.7
18	2.22	3.47	101.5	137.	76.3
19	2.37	3.48	107.5	134	73.8
20	2.40	3.46	109.5	96	79.2
21	2.33	3.45	106.5	52	77.3
22	2.13	3.49	97.0	54	78.5

Card Data.

Run 2. Right High Pressure Cylinder.

Crank End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in $\frac{1}{2}$	I.H.P.	% cut off
1	1.90	3.41	89.0	123	66.2
2	2.13	3.50	96.8	128	61.5
3	2.12	3.48	97.0	121	61.8
4	2.24	3.44	103.2	125	65.2
5	2.30	3.44	105.8	114	65.9
6	2.39	3.45	109.3	127	69.6
7	2.39	3.51	107.5	123	69.8
8	2.17	3.52	98.0	138	70.1
9	2.17	3.48	99.0	153	73.5
10	2.09	3.46	97.0	150	65.7
11	2.08	3.50	94.7	134	62.4
12	2.28	3.46	104.3	133	69.5
13	2.48	3.48	112.3	132	69.0
14	2.17	3.46	99.5	131	64.5
15	2.08	3.47	95.3	144	65.0
16	2.25	3.46	103.0	149	76.5
17	2.09	3.50	95.0	115	66.4
18	2.07	3.47	95.0	125	68.4
19	2.09	3.48	95.4	106	61.8
20	2.26	3.46	103.4	84	73.5
21	2.22	3.45	102.0	48	72.1
22	2.06	3.49	94.0	51	71.8

Card Data.

Run 3. Left Low Pressure Cylinder.					
No. of card	Area in sq. in.	Length in in.	MFE.P. in #	Head End. I.H.P.	% out off
1	2.42	3.77	39.2	139	62.9
2	2.21	3.68	35.7	133	74.3
3	---	---	---	---	---
4	---	---	---	---	---
5	---	---	---	---	---
6	---	---	---	---	---
7	---	---	---	---	---
8	---	---	---	---	---
9	2.81	3.73	44.7	121	72.7
10	2.71	3.73	43.2	161	70.3
11	2.38	3.72	38.0	128	76.4
12	2.61	3.71	41.8	151	76.6
13	2.64	3.72	42.2	139	66.5
14	2.80	3.71	45.0	99	74.4
15	---	---	---	---	---
16	3.15	3.67	51.9	78	75.0
17	2.85	3.66	47.0	72	77.0
18	3.00	3.67	48.5	58	69.7
19	2.93	3.68	47.3	72	76.7
20	2.52	3.73	40.2	122	72.4
21	---	---	---	---	---
22	2.87	3.70	46.2	144	46.2
23	---	---	---	---	---
24	---	---	---	---	---
25	3.08	3.67	50.0	178	74.8
26	3.45	3.67	40.8	97	80.0
27	2.80	3.63	45.8	70	76.5
28	---	---	---	---	---

Card Data.					
Run 3. Left Low Pressure Cylinder.	Crank End.				
No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% out off.
1	2.54	3.77	40.0	140	70.4
2	2.38	3.68	38.3	141	72.7
3	---	---	---	---	---
4	---	---	---	---	---
5	---	---	---	---	---
6	---	---	---	---	---
7	---	---	---	---	---
8	---	---	---	---	---
9	2.95	3.73	47.0	125	70.4 47.0
10	2.68	3.73	42.7	157	75.4 42.7
11	2.49	3.72	59.7	133	71.6
12	2.67	3.71	43.0	154	75.4
13	2.68	3.72	42.8	139	73.0
14	2.61	3.71	42.8	98	74.4
15	---	---	---	---	---
16	3.25	3.67	52.8	80	75.0
17	2.69	3.66	43.7	66	74.7
18	2.99-	3.67	48.5	57	73.5
19	3.21	3.68	52.0	78	78.0
20	2.52	3.73	40.3	121	74.8
21	---	---	---	---	---
22	2.90	3.70	46.5	143	73.5 46.5
23	---	---	---	---	---
24	---	---	---	---	---
25	3.15	3.67	51.0	178	85.0
26	3.46	3.65	40.0	94	87.2
22	3.13	3.63	51.3	77	71.9
28	---	---	---	---	---

Card Data.						79
Run 3. Right Low Pressure Cylinder.				Head End.		
No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off	
1	2.30	3.56	38.3	136	75.0	
2	2.16	3.54	36.2	135	74.4	
3	2.30	3.57	38.2	136	74.9	
4	2.44	3.54	41.0	125	69.3	
5	2.47	3.55	41.3	94	75.7	
6	2.44	3.33	43.8	52	78.7	
7	2.71	3.53	45.5	93	78.3	
8	2.46	3.57	41.0	104	76.7	
9	2.62	3.56	43.7	118	75.3	
10	2.52	3.59	41.7	155	78.0	
11	2.46	3.57	41.0	139	80.2	
12	2.61	3.55	43.7	158	68.4	
13	2.49	3.56	41.5	137	75.0	
14	2.28	3.55	38.0	83	77.1	
15	2.33	3.52	39.3	80	75.9	
16	2.84	3.49	48.3	74	74.8	
17	2.89	3.50	49.2	75	78.5	
18	2.72	3.52	46.0	55	68.7	
19	2.69	3.49	45.8	70	77.3	
20	2.52	3.55	42.2	128	78.9	
21	2.48	3.56	41.3	146	77.5	
22	2.61	3.53	44.0	137	71.7	
23	2.72	3.55	45.5	142	77.8	
24	2.33	3.55	39.0	154	77.5	
25	2.58.	3.56	44.7	159	74.7	
26	3.04	3.50	51.7	122	74.6	
27	2.82	3.54	47.3	72	73.5	
28	2.80	3.73	38.2	119	68.1	

Card Data.					
Run 3. No. of card	Right Low Pressure Cylinder. Area in sq. in.	Length in in.	M.E.P. in #	Crank End. I.H.P.	% out off.
1	2.43	3.56	40.5	142	71.4
2	2.22	3.54	37.2	137	71.5
3	2.22	3.57	36.9	129	70.9
4	2.61	3.54	44.0	132	71.5
5	2.43	3.55	40.7	92	63.3
6	1.87	3.33	33.2	39	66.2
7	2.53	3.53	42.5	85	76.7
8	2.25	3.57	37.4	94	70.2
9	2.62	3.56	41.8	112	74.5
10	2.43	3.59	40.3	148	73.0
11	2.38	3.57	39.7	133	73.7
12	2.47	3.55	41.3	148	74.1
13	2.29	3.56	38.2	125	77.0
14	2.49	3.55	42.7	93	74.8
15	2.20	3.52	37.0	74	73.0
16	2.95	3.49	51.5	78	77.4
17	2.76	3.50	47.0	71	74.8
18	2.78 $\frac{1}{2}$	3.52	47.0	55	66.0
19	2.73	3.49	46.6	70	76.1
20	2.23	3.55	47.3	112	76.3
21	2.32	3.56	38.7	136	75.0
22	2.70	3.53	45.5	140	75.4
23	2.69	3.55	45.2	139	75.4
24	2.30	3.55	38.5	151	76.1
25	2.58	3.56	43.0	151	73.3
26	3.07	3.50	52.2	122	73.5
27	2.87	3.54	48.3	78	74.6
28	2.30	3.73	36.5	113	65.8

Card Data.

81

Run 3. Left High Pressure Cylinder. Head End.					
No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	2.30	3.57	101.3	141	165.2
2	2.17	3.53	97.7	146	87.2
3	2.13	3.53	96.4	125	67.4
4	2.20	3.55	98.4	122	66.2
5	2.21	3.57	98.1	89	70.0
6	2.17	3.53	97.8	58	72.4
7	2.68	3.55	118.5	99	55.70
8	2.49	3.56	110.5	113	59.45
9	2.67	3.61	116.4	126	69.5
10	2.35	3.56	104.7	152	70.2
11	2.32	3.60	102.4	128	70.8
12	2.16	3.60	95.4	135	67.5
13	2.22	3.54	99.4	100	68.7
14	2.57	3.56	113.8	91	68.6
15	2.46	3.51	111.0	85	69.5
16	2.58	3.60	113.1	84	69.5
17	2.60	3.55	115.4	68	70.5
18	2.29	3.53	102.4	50	68.0
19	2.22	3.57	103.8	48	72.9
20	2.28	3.57	103.0	119	69.0
21	2.58	3.60	113.0	157	70.0
22	2.45	3.62	106.8	129	70.0
23	2.16	3.61	95.0	119	69.0
24	2.47	3.62	108.2	181	69.3
25	2.37	3.54	118.8	165	69.2
26	2.53	3.48	114.4	114	71.1
27	---	---	---	---	---
28	---	---	---	---	---

Card Data.

Run 3. No. of card.	Left High Area in sq. in.	Pressure. Length in in.	Crank End. M.E.P. in #	I.H.P.	% cut off.
1	2.31	3.57	102.4	137	70.7
2	2.27	3.53	102.0	148	70.3
3	2.25	3.52	101.5	136	70.0
4	2.16	3.55	96.7	117	71.5
5	2.38	3.57	105.5	93	74.0
6	2.35	3.53	108.4	61	76.5
7	2.69	3.55	113.2	92	73.8
8	2.60	3.56	115.4	113	71.4
9	2.54	3.61	111.4	116	74.8
10	2.45	3.55	109.0	154	73.0
12	2.35	3.60	103.4	126	73.3
12	2.31	3.60	102.0	141	74.5
13	2.30	3.54	103.0	135	71.2
14	2.54	3.56	113.0	88	75.0
15	2.48	3.51	111.4	83	72.4
16	2.64	3.60	116.0	82	72.6
17	2.42	3.55	108.4	62	74.4
18	2.43	3.53	110.5	57	66.5
19	2.16	3.57	96.5	46	76.8
20	2.35	3.57	97.5	111	71.7
21	2.57	3.60	113.0	152	72.3
22	2.55	3.62	111.5	130	74.5
23	2.44	3.61	106.5	129	74.5
24	2.57	3.62	112.5	189	74.6
25	2.24	3.54	101.0	135	74.9
26	2.52	3.48	114.5	100	74.7
27	---	---	---	---	---
28	---	---	---	---	---

Card Data.					
Run 3.	Right High Pressure	Cylinder.	Head End.		
No. of	Area in	Length	M.E.P.	I.H.P.	% cut
card	sq. in.	in in.	in #		off
1	2.25	3.42	103.7	139	70.3
2	2.00	3.50	91.2	132	69.3
3	2.20	3.45	101.0	135	70.5
4	2.12	3.42	98.5	119	67.0
5	2.17	3.42	102.0	89	73.0
6	2.10	3.38	98.7	57	78.4
7	2.60	3.43	118.5	96	74.4
8	2.49	3.42	114.5	112	74.5
9	2.43	3.44	111.5	116	71.5
10	2.34	3.43	107.5	152	73.9
11	2.26	3.46	103.4	126	74.4
12	2.30	3.50	104.0	144	74.5
13	2.17	3.46	98.7	129	73.7
14	2.44	3.43	112.5	88	74.1
15	2.45	3.43	113.5	84	73.7
16	2.38	3.39	110.5	79	71.4
17	2.43	3.43	111.5	64	76.4
18	2.43	3.42	112.0	53	75.8
19	2.36	3.43	108.5	51	74.6
20	2.15	3.46	98.7	113	73.4
21	2.19	3.46	101.2	136	74.0
22	2.43	3.43	111.7	131	71.8
23	2.34	3.50	105.7	144	71.7
24	2.56	3.41	121.5	178	71.5
25	2.61	3.44	118.8	158 1/2	75.1
26	2.66	3.41	121.5	112	82.9
27	2.69	3.43	122.5	66	71.2
28	2.18	3.46	100.0	124	75.2

Card Data.

Run 3. No. of card	Right High Area in sq. in.	Pressure Length in in.	Cylinder. M.E.P. in #	Crank I.H.P.	End 84 % cu off
1	2.08	3.42	96.5	134	65.1
2	2.01	3.50	91.5	136	65.7
3	1.99	3.45	92	128	65
4	2.01	3.42	93.7	117	64.4
5	2.04	3.40	96.5	87	68
6	2.07	3.38	98.0	58	70.7
7	2.38	3.43	109.5	92	66.2
8	2.34	3.42	108.5	110	66.64
9	2.22	3.44	102.5	111	65.5
10	2.19	3.43	99.8	145	66.2
21	2.19	3.46	95.5	119	68
22	2.02	3.50	92.0	131	60.5
13	2.13	3.46	97.8	99	65
14	2.35	3.43	108.5	87	67.5
15	2.31	3.43	107.2	82	68
16	2.32	3.39	108.5	79	67.3
17	2.25	3.43	103.8	61	66.8
18	2.31	3.42	105.5	51	70.3
19	2.25	3.43	104	51	70.3
20	1.97	3.46	91.2	108n	67.3
21	2.14	3.46	98.5	137	67.3
22	2.15	3.43	99.5	120	66.2
23	2.33	3.44	107.5	135	65.5
24	2.03	3.50	92.5	161	64.4
25	2.18	3.41	101.5	141	67.5
26	2.47	3.41	113.5	103	69.4
27	2.50	3.43	114.5	64	67.1
28	2.01	3.46	92.7	119	65

Card Data.

Run 4, Left Low Pressure Cylinder.

Head End.

No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off
1	1.75	3.70	28.0	142	66.7
2	1.69	2.65	23.3	145	64.2
3	1.58	3.30	28.2	119	70.4
4	1.76	3.74	27.8	92	70.9
5	1.51	3.74	27.8	49	70.9
6	2.26	3.71	36.7	81	70.6
7	2.15	3.71	34.3	104	65.2
8	1.92	3.73	30.5	139	66.2
9	1.70	3.77	26.7	129	67.0
10	1.68	3.73	26.7	113	68.1
11	1.64	3.72	26.0	86	68.6
12	1.68	3.73	26.7	61	72.4
13	1.62	3.71	25.8	33	79.8

Card Data.

Run 4. Left Low Pressure Cylinder.

Crank End.

No. of card	Area in sq in.	Length in in.	M.E.P. in #	I.H.P.	% out off
1	1.79	3.70	28.0	140	66.7
2	1.69	3.65	27.3	148	64.2
3	1.58	3.30	28.2	113	70.4
4	1.76	3.74	27.8	79	70.9
5	1.51	3.74	27.8	46	70.9
6	2.17	3.71	34.7	67	72.0
7	2.02	3.71	32.2	91	65.5
8	1.92	3.73	30.5	130	66.2
9	1.72	3.72	27.0	119	68.2
10	1.71	3.73	27.0	103	71.1
11	1.72	3.72	27.3	87	70.5
12	1.79	3.73	28.3	59	77.3
13	1.52	3.71	21.8	22	21.8

Card Data.

Run 4. Right Low Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% Cut off.
1	1.71	3.56	28.3	143	71.7
2	1.59	3.60	26.0	139	69.5
3	1.53	3.57	25.2	106	71.9
4	1.52	3.55	25.2	83	66.2
5	1.64	3.53	27.3	56	72.5
6	1.51	3.55	25.2	56	69.6
7	1.98	3.55	33.2	102	67.1
8	1.91	3.56	31.8	145	67.5
9	1.69	3.57	28.0	135	67.9
10	1.66	3.56	27.5	116	69.7
11	1.57	3.54	26.2	87	74.0
12	1.54	3.57	35.5	58	77.2
13	1.51	3.55	25.0	32	25.0

Card Data.

Run 4. Right Low Pressure Cylinder.

Crank End.

No. of card	Area in sq. in.	Length in in.	$\frac{1}{2}$ in #	M.E.P. in #	I.H.P.	% out off.
1	1.60	3.56		26.5	133	68.0
2	1.39	3.60		22.7	123	64.1
3	1.51	3.57		25.0	100	67.2
4	1.39	3.55		23.0	65	66.7
5	1.51	3.53		25.2	49	72.2
6	1.41	3.55		23.3	45	68.0
7	1.80	3.55		30.0	85	67.1
8	1.82	3.56		30.2	128	66.9
9	1.53	3.57		25.2	111	66.5
10	1.59	3.56		26.3	101	68.2
11	1.60	3.54		26.7	85	71.2
12	1.51	3.57		25.0	52	75.1
13	1.48	3.55		24.5	25	74.6

Card Data.

Run 4. Left High Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in $\frac{1}{2}$	I.H.P.	% cut off.
1	1.73	3.61	77.5	161	61.5
2	1.51	3.58	69.0	156	60.6
3	1.50	3.57	68.7	115	61.6
4	1.48	3.60	67.2	80	60.3
5	1.47	3.58	67.2	54	64.2
6	---	---	---	---	---
7	1.97	3.50	90.0	106	60.6
8	1.74	3.55	79.0	140	60.0
9	1.62	3.58	73.5	135	62.4
10	1.55	3.57	70.7	113	63.5
11	1.50	3.59	68.0	90	64.1
12	1.62	3.54	74.3	65	67.9
13	1.54	3.57	70.5	29	69.0

Card Data.

Run 8. Left High Pressure Cylinder.

Crank End.

No. of card.	Area in sq. in.	Length in in.	M.E.P. in #	I.H.P.	% cut off.
1	1.89	3.61	84.2	170	66.5
2	1.67	3.58	75.5	165	61.7
3	1.61	3.57	73.3	118	63.8
4	1.61	3.60	72.7	83	67.2
5	1.57	3.58	71.3	56	63.5
6	---	---	---	---	---
7	2.20	3.50	99.7	114	64.7
8	1.96	3.55	94.2	162	62.1
9	1.67	3.58	75.5	134	64.4
10	1.65	3.57	75.0	116	67.2
11	1.60	3.54	73.5	94	66.9
12	1.54	3.57	75.5	62	72.5
13	1.55	3.57	75.6	31	75.3
14					

Card Data.

Run 4. Right High Pressure Cylinder.

Head End.

No. of card	Area in sq. in.	Length in in.	M.E.P. in#	I.H.P.	% cut off.
1	1.90	3.44	89.5	184	64.8
2	1.75	3.44	82.0	185	65.5
3	1.59	3.45	82.0	125	62.9
4	1.48	3.47	69.5	82	63.7
5	1.51	3.41	72.0	58	70.7
6	2.23	3.43	103.0	83	70.8
7	2.16	3.46	99.5	117	64.2
8	2.00	3.46	92.5	164	62.7
9	1.78	3.48	82.3	152	82.3
10	1.69	3.46	76.5	127	68.2
11	1.63	3.45	76.5	101	76.5
12	1.58	3.41	75.0	65	75.0
13	---	---	---	---	---

Card Data.

Run 4. Right High Pressure Cylinder.

Crank End.

No. of card.	Area in $\frac{1}{2}$ sq. in	Length in in.	M.E.P. in $\frac{1}{2}$	I.H.P.	% out off
1	1.68	3.44	79.0	160	57.3
2	1.49	3.44	70.4	155	57.0
3	1.23	3.45	59.0	95	55.9
4	1.12	3.47	54.0	62	58.5
5	1.41	3.41	67.7	53	65.1
6	2.13	3.43	91.5	71	57.2
7	1.96	3.46	90.5	103	57.9
8	1.65	3.45	77.0	132	55.0
9	1.60	3.48	84.5	133	74.5
10	1.65	3.46	77.0	119	62.1
11	1.53	3.45	72.0	92	65.8
12	1.60	3.41	76.0	64	68.0
13	---	---	---	---	---

I.H.P. Summary.

Run 1.

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Card.	L.P.L.		L.P.R.		Total	H.P.L.		H.P.R.		Total	Total I.H.P.
	H.E.	C.E.	H.E.	C.E.	Low Press.	H.E.	C.E.	H.E.	C.E.	High Press.	
1	131	133	123	125	512	165	174	204	159	702	1214
2	129	134	129	134	517	156	180	170	166	672	1189
3	131	136	131	136	524	145	131	145	115	536	1060
4	140	132	140	132	535	153	145	171	153	622	1157
5	131	138	127	124	520	135	137	150	136	558	1078
6	131	126	100	118	475	146	145	146	137	574	1049
7	131	131	115	117	494	143	148	160	141	592	1086
8	143	144	139	138	564	154	147	158	148	607	1171
9	137	148	116	103	505	163	151	170	138	622	1127
10	157	157	125	122	561	154	144	154	144	588	1149
11	117	121	144	125	507	168	138	168	138	606	1113
12	118	127	122	139	506	172	149	172	149	639	1145
13	143	146	110	116	525	179	179	179	179	712	1227
14	144	146	133	122	595	186	161	186	161	699	1294
15	147	149	140	128	564	201	181	201	181	757	1321
16	132	129	115	119	495	166	174	166	174	684	1179
17	131	132	129	124	516	184	163	184	163	697	1213
18	127	119	125	110	491	161	128	161	128	582	1073
19	91	97	100	91	379	119	103	119	103	452	831
20	117	110	117	97	441	138	117	138	117	503	944
21	167	158	189	170	684	193	168	193	168	718	1402
Page.	267.		248.		515.	309.		317.		626.	1141.

I.H.P. Summary.

Run 2.

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Bot Card	L. P. L.		L. P. R.		Total	H. P. L.		H. P. R.		Total	Total I. H. P.
	H. E.	C. E.	H. E.	C. E.	Low Press.	H. E.	C. E.	H. E.	C. E.	High Press.	1
1	133	128	136	137	534	118	147	151	123	539	1073
2	133	127	129	130	519	140	135	144	128	547	1066
3	123	133	142	127	525	130	137	138	121	526	1051
4	127	135	134	146	542	129	131	135	125	520	1062
5	123	110	111	116	460	119	115	130	114	478	938
6	118	124	130	121	493	132	132	146	127	537	1030
7	132	132	133	133	530	129	132	140	123	524	1054
8	144	152	154	147	597	162	150	166	138	616	1213
9	160	160	159	146	625	164	164	175	153	656	1281
10	158	154	158	147	617	160	150	175	150	635	1252
11	167	132	141	132	572	147	150	166	134	597	1169
12	142	133	126	126	527	141	141	147	133	562	1089
13	126	123	126	124	499	137	126	149	132	548	1047
14	154	151	154	158	617	154	134	155	131	557	1174
15	160	142	153	141	598	163	154	172	144	624	1220
16	153	148	147	146	594	127	158	178	149	648	1242
17	120	134	123	114	491	127	130	135	115	507	998
18	125	127	124	120	496	122	124	137	125	513	1009
19	118	121	122	118	479	141	124	134	106	486	965
20	89	92	88	85	354	88	87	96	84	355	709
21	57	56	58	49	220	51	48	52	48	199	419
22	46	50	51	37	184	48	52	54	51	205	389
Page	253		251		504	257		260		517	1021

I. H. P. Summary.

Run 3.

95

No. of Card	L. P. L.		L. P. R.		Total	H. P. L.		H. P. R.		Total	Total I. H. P.
	H. E.	C. E.	H. E.	C. E.	Low Press.	H. E.	C. E.	H. E.	C. E.	High Press.	
1	139	140	136	142	557	141	137	139	134	551	1108
2	133	141	135	137	543	146	148	132	136	562	1105
3	136	129	136	129	530	135	136	135	128	538	1068
4	125	132	125	132	514	122	117	119	117	475	989
5	94	92	94	92	372	89	93	89	87	358	730
6	52	39	52	39	182	58	61	57	58	234	416
7	93	85	93	85	356	99	92	96	92	379	735
8	104	94	104	94	396	113	113	112	110	448	844
9	121	125	118	112	476	126	116	116	111	469	945
10	161	157	155	148	521	152	154	152	145	603	1124
11	128	133	139	133	533	128	126	126	119	499	1032
12	151	154	158	148	611	135	141	144	131	551	1162
13	139	139	137	125	540	100	135	129	99	463	1003
14	99	93	83	93	368	91	88	88	87	354	722
15	80	74	80	74	308	85	83	84	82	334	642
16	78	80	74	78	310	84	82	79	79	324	634
17	72	66	75	71	284	68	62	64	61	255	539
18	58	57	55	55	225	50	57	53	51	211	436
19	72	78	70	70	290	48	46	51	51	196	486
20	122	121	128	112	483	119	111	113	108	451	934
21	146	136	146	136	564	157	152	136	137	582	1146
22	144	143	137	140	564	129	130	131	120	510	1074
23	142	139	142	139	562	119	129	144	135	527	1089
24	154	151	154	151	610	181	189	178	161	629	1239
25	178	178	159	151	666	165	135	158	141	599	1265
26	97	94	122	122	437	114	100	112	103	429	866
27	70	77	72	73	292	66	64	66	64	260	552
28	119	113	119	113	464	124	119	124	119	486	950
Average.	228.		224.		452	216.		218.		434.	887.

I. H. P. Summary.

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Run 4.

No. of Card	L. P. L.		L. P. R.		Total	H. P. L.		H. P. R.		Total	Total I. H. P.
	H. E.	C. E.	H. E.	C. E.	Low Press.	H. E.	C. E.	H. E.	C. E.	High Press.	
1	142	140	143	133	568	161	170	184	160	675	1243
2	145	148	138	123	554	156	165	185	155	661	1215
3	119	113	106	100	438	115	118	125	95	453	891
4	92	79	83	65	311	80	83	82	62	307	618
5	49	46	56	49	200	54	56	58	53	221	421
6	81	67	102	45	249	83	71	83	71	308	556
7	104	91	145	85	386	106	114	117	103	440	826
8	139	130	135	128	532	140	162	164	132	598	1130
9	129	119	116	111	494	135	134	152	133	554	1048
10	113	103	87	101	433	113	116	127	119	476	909
11	86	87	58	85	345	90	94	101	92	377	722
12	61	59	56	52	230	65	62	65	64	256	486
13	33	22	32	25	112	29	31	29	31	121	233
Average	194.		183.		377.	209.		214.		423.	800.

Reading	Time	Boiler Pressure	Pressure in Branch Pipe	Revolutions.		Miles per hr. Locomotive	High Press. I. H. P.	Low Press. I. H. P.	Total I. H. P.	Proportion of Work H. P. Cylinders	Dynamometer Car Push in #	Miles per hr. Dyn. Car.	Dyn. H. P.	Mechanical Efficiency				
1	9:25	205	—	43	45	6.60	702	512	1214	.580	60,500	5.90	950	78				
2	:28	205	—	41	42	6.25	672	517	1189	.565	59,000	5.90	930	78				
3	:31	206	190	41	39	6.05	536	524	1060	.505	64,000	5.75	983	85				
4	:34	204	180	41	40	6.10	622	535	1157	.537	63,200	5.75	970	84				
5	:37	201	180	39	37	5.72	558	520	1078	.515	65,000	5.40	935	87				
6	:40	202	185	37	35	5.40	574	475	1049	.545	68,500	5.00	912	87				
7	:43	207	190	41	37	5.86	592	494	1086	.550	67,500	5.00	900	83				
8	:46	200	180	44	39	6.25	607	564	1171	.515	66,500	5.69	1010	86				
9	:49	207	180	43	41	6.32	622	505	1127	.550	64,000	6.10	1040	92				
10	:52	202	190	45	40	6.40	588	561	1149	.517	64,000	6.10	1040	91				
11	:55	200	194	38	41	5.95	606	507	1113	.545	62,000	5.90	975	88				
12	:58	198	195	38	42	6.00	639	506	1145	.554	64,300	5.90	1018	89				
13	10:01	210	195	41	42	6.25	712	525	1227	.580	67,500	6.00	1080	88				
14	:04	210	185	44	44	6.60	699	595	1294	.540	68,000	6.10	1100	85				
15	:07	205	180	46	47	7.00	757	564	1321	.570	63,000	6.28	1058	80				
16	:10	205	190	43	43	6.50	684	495	1179	.580	63,500	6.50	1100	93				
17	:13	207	185	43	43	6.50	697	516	1213	.575	65,500	5.90	1030	85				
18	:16	205	182	38	37	5.65	582	491	1073	.545	65,000	5.65	980	91				
19	:19	205	180	30	30	4.50	452	379	831	.550	63,700	3.80	645	78				
20	:22	202	180	33	31	4.82	503	441	944	.510	64,600	4.00	690	73				
21	:25	198	180	52	47	7.45	718	684	1402	.510	69,500	6.00	1100	79				

Run 2.

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	Time	Boiler Pressure	Pressure in Branch Pipe	Revolutions Front Eng.	Revolutions Back Eng.	Miles per hr. Locomotive.	High Press. I. H. P.	Low Press. I. H. P.	Total I. H. P.	Work done per hour by engine.	Work done by car.	Miles per hour by car.	High Press. Dyn. Eff.	Neck. Eff. %				
1	3:47	208	175	43	41	6.50	539	534	1073	.505	58,000	6.10	940	87				
2	:50	205	175	42	39	6.30	547	519	1066	.512	62,000	5.80	970	91				
3	:53	200	180	41	37	6.00	526	525	1051	.505	62,000	5.75	970	92				
4	:56	208	175	39	36	6.00	520	542	1062	.485	65,000	4.90	850	80				
5	:59	201	180	34	32	5.40	478	460	938	.510	67,000	4.75	850	91				
6	4:02	205	185	35	40	4.75	537	493	1030	.525	70,000	4.75	900	87				
7	:05	209	180	38	34	5.50	524	530	1054	.505	69,000	5.50	1007	95				
8	:08	207	180	48	42	6.75	616	597	1213	.510	64,000	6.25	1160	96				
9	:11	202	170	50	46	7.60	656	625	1281	.510	64,000	7.25	1200	94				
10	:14	200	170	49	46	7.50	635	617	1252	.510	62,000	6.75	1100	88				
11	:17	202	165	45	42	6.60	597	572	1169	.510	62,000	5.75	970	83				
12	:20	202	175	40	38	5.90	562	527	1089	.515	66,000	5.25	915	84				
13	:23	210	185	36	35	5.50	548	499	1047	.520	70,000	5.00	950	91				
14	:26	205	175	42	36	6.00	557	617	1174	.470	65,000	6.00	1075	92				
15	:29	205	185	49	45	7.50	624	598	1220	.510	59,000	7.20	1171	96				
16	:32	205	175	46	43	6.85	648	594	1242	.520	64,000	5.80	1000	81				
17	:35	205	160	39	36	6.00	507	491	998	.507	59,000	5.65	885	89				
18	:38	177	155	41	39	6.10	513	496	1009	.510	60,000	5.65	850	84				
19	:41	175	160	39	36	5.80	486	479	965	.504	59,000	4.35	690	72				
20	:44	175	160	27	24	4.25	355	354	709	.500	64,000	2.40	410	58				
21	:47	170	160	17	14	2.20	199	220	419	.475	65,000	1.90	333	79				
22	:50	155	160	18	16	2.75	205	184	389	.525	55,000	2.50	368	95				

Рыцз.

Time	Boiler Pressure	Pressure Branch Pipe	Revolutions Front Eng.	Revolutions Back Eng.	Miles per hr. Locomotive	High Press. I. H. P.	Low Press. I. H. P.	Total I. H. P.	Proportion of High Press. Cyl.	Dyn. Car Push in #.	Miles per Dyn. Car.	Dyn. H. P.	Mech. Eff. %.			
11:18	185	175	42	40	6.12	561	557	1108	.497	60,000	5.85	935	84			
:21	187	175	44	43	6.50	562	543	1105	.508	59,000	6.05	952	86			
:24	183	170	42	40	6.12	538	530	1068	.505	58,000	6.05	932	87			
:27	177	168	36	36	5.38	475	514	989	.480	60,000	5.25	840	85			
:30	170	163	27	26	3.96	358	372	730	.490	61,500	3.51	565	77			
:33	165	160	14	17	2.17	234	182	416	.562	62,500	2.00	332	80			
:47	200	190	24	24	3.60	379	356	735	.515	67,000	3.30	590	80			
:50	197	190	30	29	4.50	448	396	844	.531	65,000	4.47	775	92			
:53	195	180	32	31	5.50	469	476	945	.496	63,000	5.30	890	94			
:56	195	175	44	42	6.45	603	521	1124	.463	61,500	5.85	960	85			
:59	190	170	40	36	5.65	499	533	1032	.515	61,500	4.95	810	78			
12:02	187	173	43	41	6.30	551	611	1162	.525	59,000	5.45	860	77			
:05	180	160	39	39	5.72	403	540	1003	.538	63,500	3.48	587	58			
:29	188	180	26	23	3.65	354	368	722	.490	67,000	2.90	537	74			
:32	188	186	24	22	3.42	334	308	642	.520	63,000	2.33	390	61			
:35	195	185	18	21	2.91	324	310	634	.511	66,000	2.33	416	66			
:38	190	180	18	17	2.77	255	284	539	.474	66,000	1.85	325	60			
:41	185	177	14	14	2.10	211	225	436	.484	64,000	1.85	310	71			
:44	182	175	18	14	2.40	196	290	486	.403	64,000	1.85	310	64			
:47	182	165	36	34	5.31	451	483	934	.484	60,000	5.30	850	91			
:50	192	172	42	40	6.15	582	564	1146	.491	61,000	5.30	862	75			
:53	200	185	37	35	5.40	510	564	1074	.524	65,000	5.04	860	80			
:56	207	190	37	36	5.45	527	562	1089	.515	63,000	5.42	920	85			
:59	210	173	47	50	7.25	629	610	1239	.507	60,000	6.40	1020	83			
1:02	207	185	42	40	6.30	599	666	1265	.472	64,000	5.67	1024	81			
:05	205	195	28	26	4.26	429	437	866	.495	70,000	3.50	1009	79			
:08	198	184	18	16	2.55	260	292	552	.471	70,000	2.25	420	76			
:11	193	160	37	37	5.80	485	464	950	.511	60,000	5.67	905	95			

Run 4.

Mileage	Time	Boiler Pressure.	Pressure in Branch Pipe.	Revolutions. Front Eng.	Revolutions. Back Eng.	Miles per hr. Locomotive.	High Press. I. H. P.	Low Press. I. H. P.	Total I. H. P.	Proportion of Work. High Press. Cyl.	Dyn. Car Pushing #.	Miles per hr. Dyn. Car.	Dyn. H.P.	Mech. Eff. %.			
1	5:00	170	145	60	60	9.00	675	568	1243	.542	44,000	6.85	805	65			
2	:03	160	135	63	65	9.60	661	554	1215	.544	38,000	8.25	830	68			
3	:06	150	125	50	48	7.35	453	438	891	.507	36,500	6.85	670	75			
4	:09	135	120	39	34	5.50	307	311	618	.496	37,500	4.00	400	65			
5	:12	128	112	24	23	3.62	221	200	421	.525	37,500	3.50	350	83			
6	:47	180	163	26	23	3.70	308	249	556	.554	47,000	3.50	500	90			
7	:50	175	157	36	34	5.25	440	386	826	.532	48,000	4.60	590	72			
8	:53	180	155	54	51	7.90	598	532	1130	.530	45,000	7.80	910	81			
9	:56	155	135	57	53	8.30	554	494	1048	.482	39,500	7.80	820	78			
10	:59	145	125	50	46	7.20	476	433	909	.524	39,000	6.65	690	76			
11	6:02	135	120	37	38	5.30	377	345	722	.522	38,000	4.90	500	69			
12	:05	135	112	27	25	3.90	256	230	486	.526	36,000	3.90	375	77			
13	:08	120	107	15	12	2.02	121	112	233	.520	37,000	1.50	148	64			

collection of Philip M. Goldstein

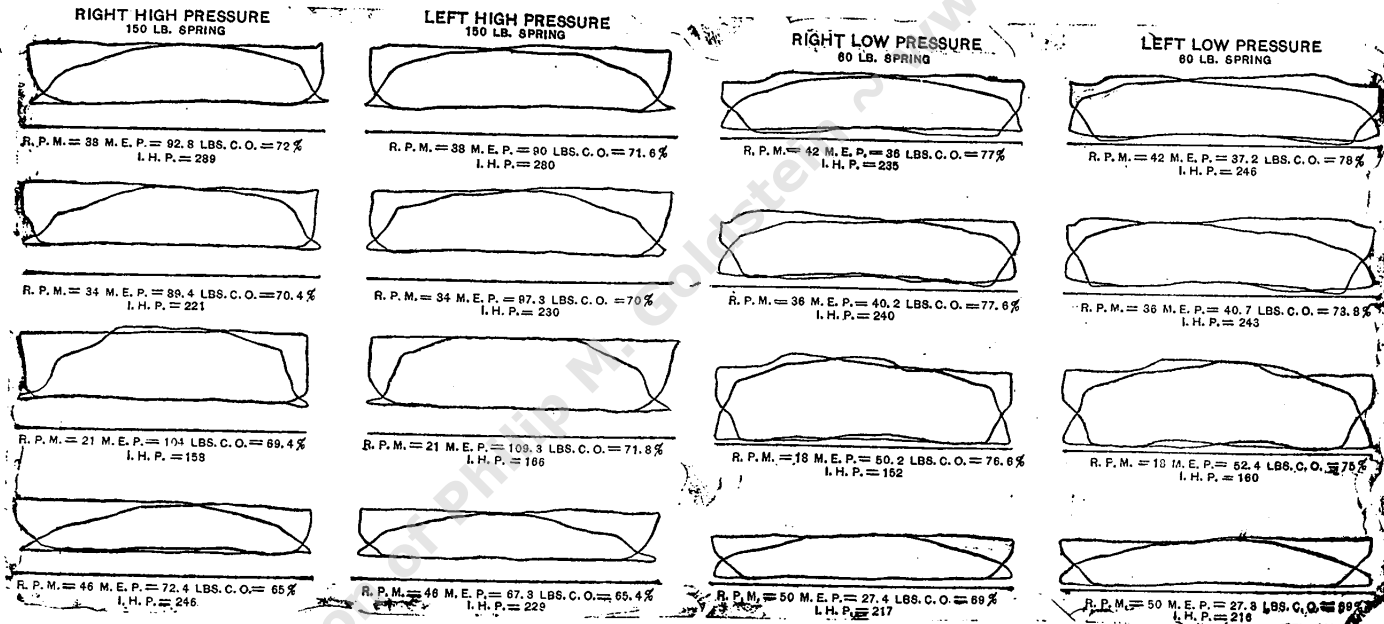
Summary.

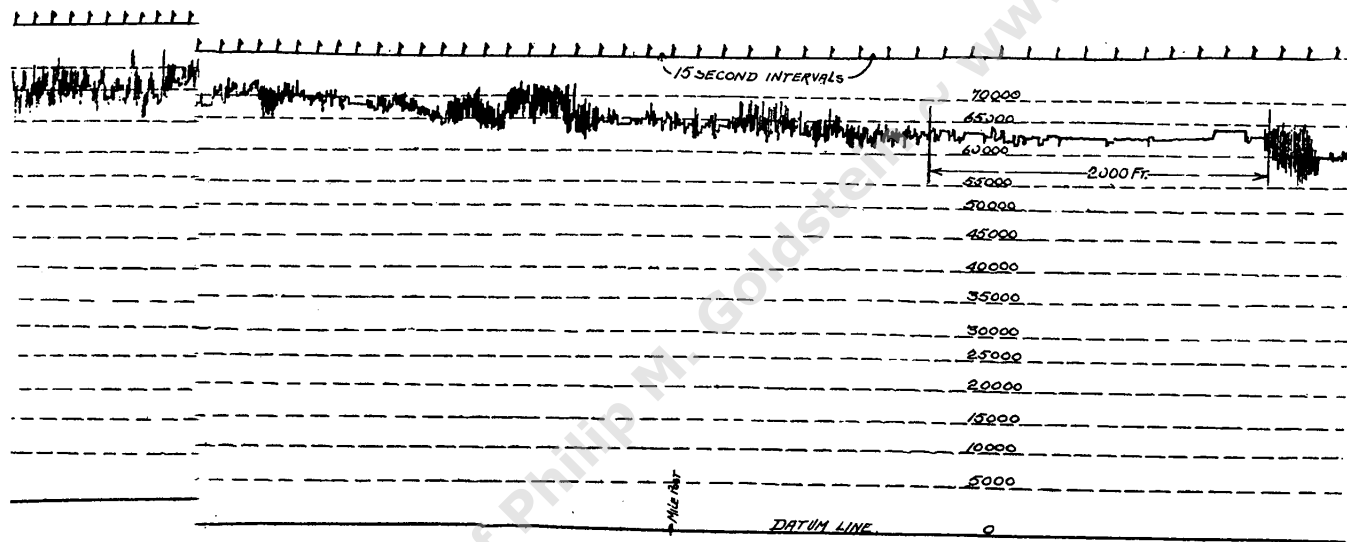
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Run No.	Time in Min.	Front Engine.							Rear Engine							Pressure in #/sq in							Drafting of water by smoke box and dynamometer injector	Min. action	Temp. of Feed Water	Lbs. Water.			Dynamometer Push in lbs.		
		R. P. M.			Aver. Speed miles per hr.	Piston Speed, ft/min			R. P. M.			Aver. Speed miles per hr.	Piston speed ft/min			In boiler.			In branch pipe	In receiver	Barometer in Hg.	Delivered to Injector				Leak. and Loss.	Deliv. and Evap.	Aver.	Max.	Min.	
		Max.	Min.	Aver.		Max.	Min.	Aver.	Max.	Min.	Aver.		Max.	Min.	Aver.	Max.	Min.														
1	60	52	30	40	6.0	244.0	141.0	187.0	47	30	40	6.0	220.0	141.0	187.0	200	210	198	185	—	28.93	4.7	53	37.7	43.410	505	42.905	64,000	69,500	59,000	
2	63	50	17	40	6.0	233.5	79.0	187.0	51	14	40	6.0	240.0	66.0	187.0	200	210	155	171	49.3	28.77	5.0	58	37.8	40.270	552	39.718	66,000		59,000	
3	113	47	14	33	5.0	220.0	66.0	154.0	50	14	33	5.0	233.5	66.0	154.0	195	210	165	176	59.5	29.06	4.5	99	37.5	48.920	942	47.978	64,000	70,000	58,000	
4	68	63	15	43	6.5	292.0	77.0	200.0	65	12	43	6.5	302.0	56.0	200.0	160	180	120	132	46.6	28.74	3.0	56	38.6	20,540	532	20,008	41,000	48,000	36,000	

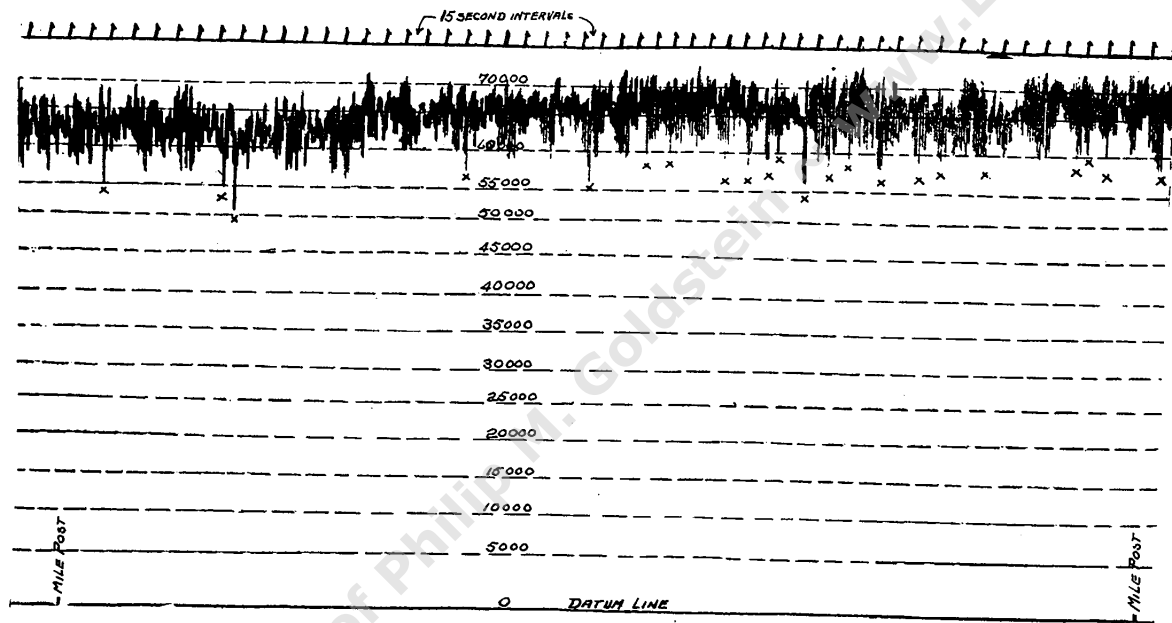
Run No.	Coal.													Flue Gas.				Evaporation in lbs.			Equivalent Evaporation from and at 212°					Boiler.		Temp. of Atmosphere		
	Coal Fired			Total			Analysis of coal by %			Coal Value B.T.U.		Dry Coal, lbs.		% O	% CO	% CO ₂	% N	Steam per hr.			Dry Steam lb. of dry coal.	Per hr.	Per lb. of coal.	Per lb. of coal			B.H.P.		Eff.	
	Kind.	Lbs. Total	% Moist	Dry Coal Fired	Combust. at 212°	Ash by analysis	Fixed C	Volatile matter	Moist %	Ash %	Of Moist Coal	Of Combust.	Per hr.					Per G.S.	Moist	Dry				Dry 8° H.S.	As Fired	Dry				Combustible
1	Bit.	5065	3.1	4910	4125	785	52.8	28.4	3.1	15.7	12100	14910	4910	4910	—	—	—	—	42905	42500	7.96	8.66	52300	9.85	10.46	10.70	12.70	152.0	80.5	40
2	"	3711	3.1	3595	3050	545	52.8	28.4	3.1	15.7	12100	14910	3520	3520	4.08	0.36	12.68	82.67	37900	37065	6.95	10.02	45800	8.15	11.50	13.00	15.50	1330	—	42
3	"	7212	3.1	7000	5900	1100	52.8	28.4	3.1	15.7	12100	14910	3980	3980	8.86	1.12	9.68	80.34	25460	25060	4.70	6.32	31000	5.84	7.80	8.08	9.60	900	61.0	31
4	"	3986	3.1	3867	3867	607	52.8	28.4	3.1	15.7	12100	14910	3415	3415	6.37	0.40	10.50	82.73	17700	17440	3.28	5.12	21400	4.04	6.10	6.22	7.40	625	48.0	46

Run No.	I. H. P.								Division of Power.								Locomotive.												Mech. Eff.	Est. of Loco.
	R. H. P.		L. H. P.		R. L. P.		L. L. P.		H. P.		L. P.		Right Side	Left Side	Total I. H. P.	Consumed per I. H. P. Hr.			Dynam. H. P.	Lbs. per R. H. P. Hr.		B. T. U. per R. H. P. Hr.	I. H. P. per %		D. H. P. per %		Machine Friction in H. P.	Draw- bar Push.		
	H. E.	C. E.	H. E.	C. E.	H. E.	C. E.	H. E.	C. E.	Right Side	Left Side	Right Side	Left Side				Lbs. of Dry Coal	Lbs. of Dry Steam	B. T. U.		Dry Coal	Dry Steam		H. S.	G. S.	H. S.	G. S.				
1	166	151	153	156	127	121	129	138	317	309	248	267	564	577	1141	4.30	37.5	53,700	97.2	5.14	23.9	66,800	.215	112.1	.169	82.9	169	14,400	75.8	3.8
2	129	128	129	122	127	124	127	126	260	257	251	253	511	510	1021	3.45	36.5	43,200	898.6	3.92	41.4	49,000	.192	102.1	.169	82.9	122	7,600	87.1	5.1
3	108	110	109	107	113	111	111	117	218	216	224	228	442	444	887	4.50	29.0	56,400	694.4	5.25	36.0	65,500	.167	82.7	.131	62.4	195	14,600	78.5	2.0
4	113	110	102	106	96	88	98	95	214	208	183	194	397	403	800	4.30	22.0	53,700	584.0	5.85	30.0	73,000	.151	80.0	.110	52.4	216	12,400	73.0	3.1

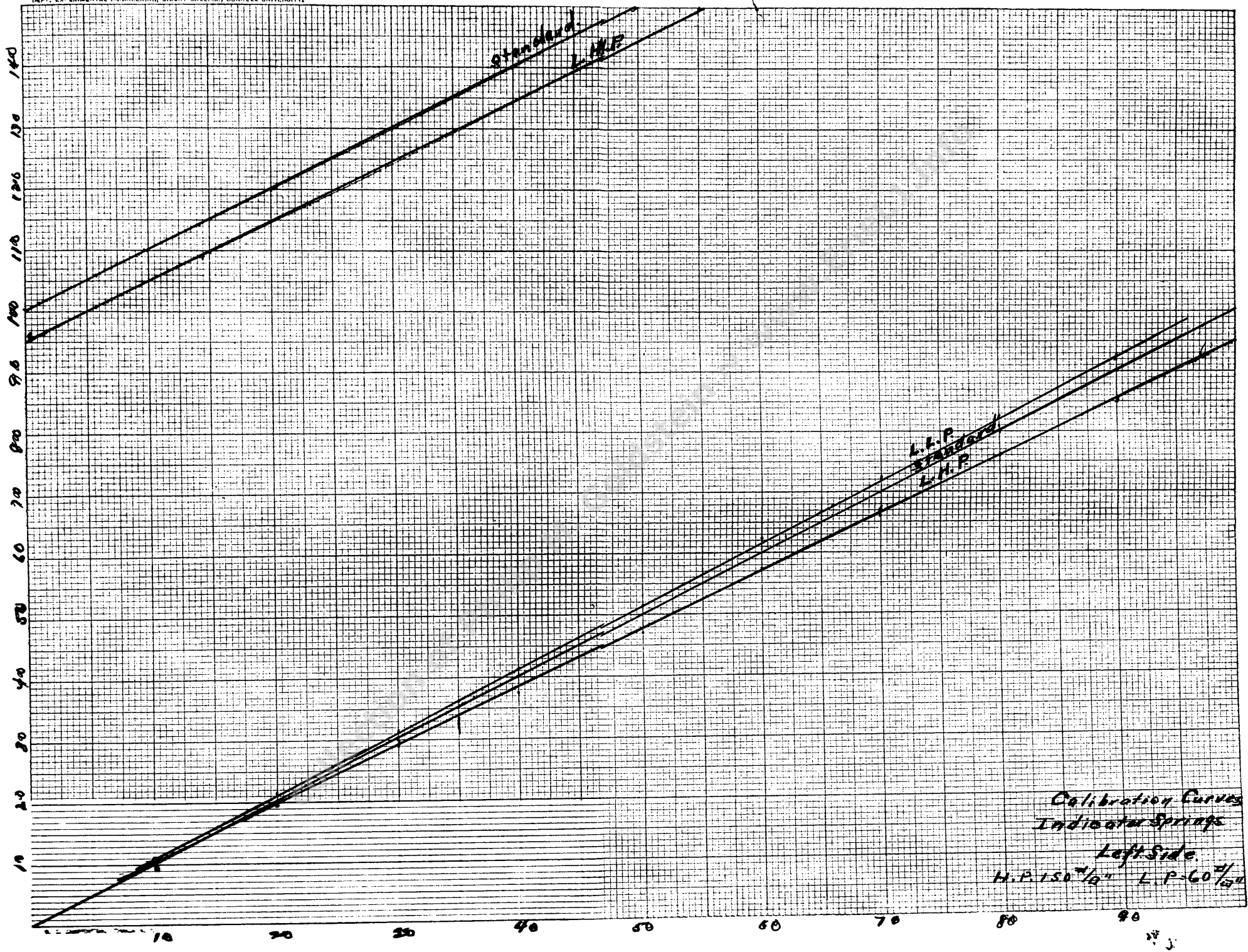




Sample Dynamometer Record, #1.



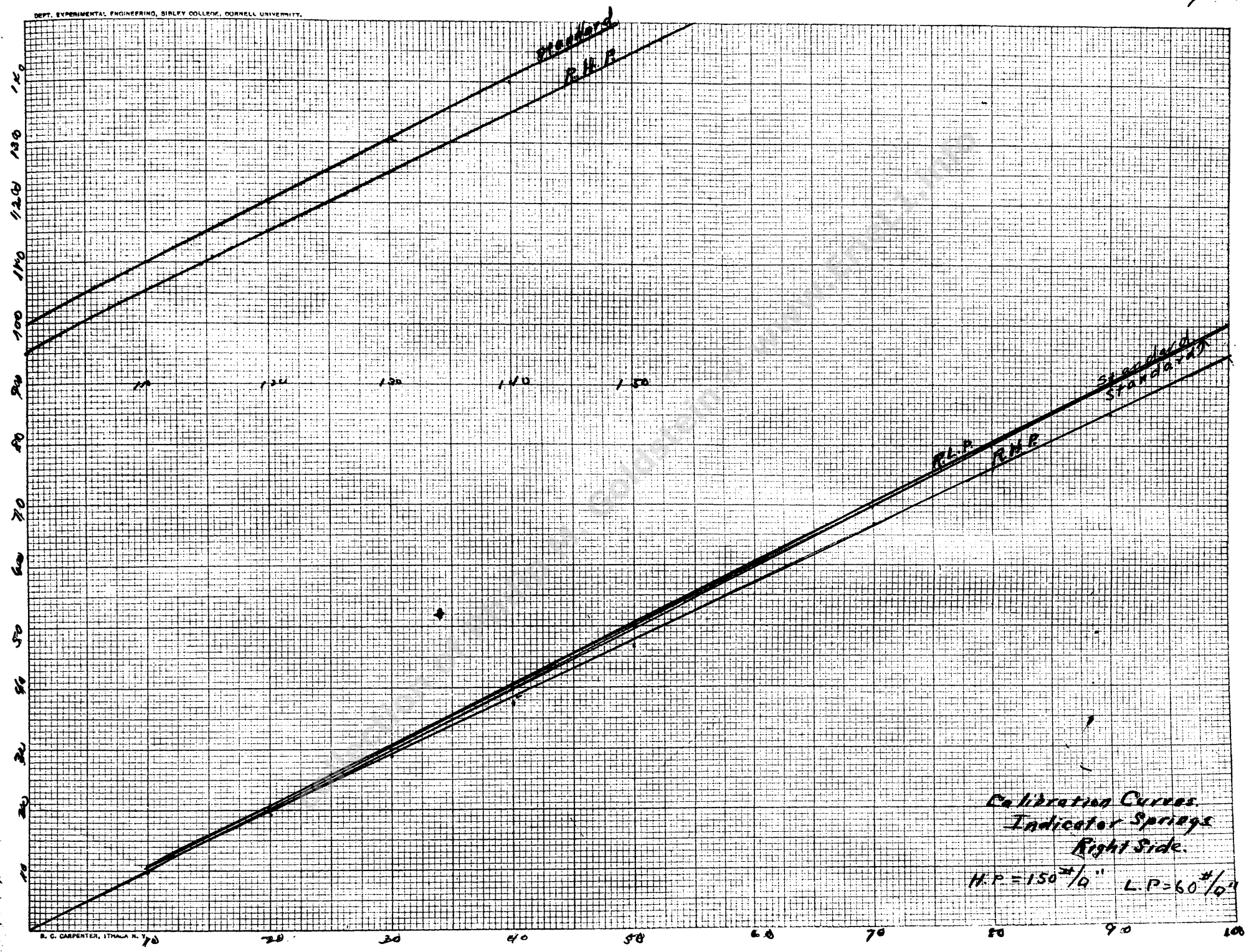
Sample Dynamometer Record, #2.
"x" indicates points of slipping.

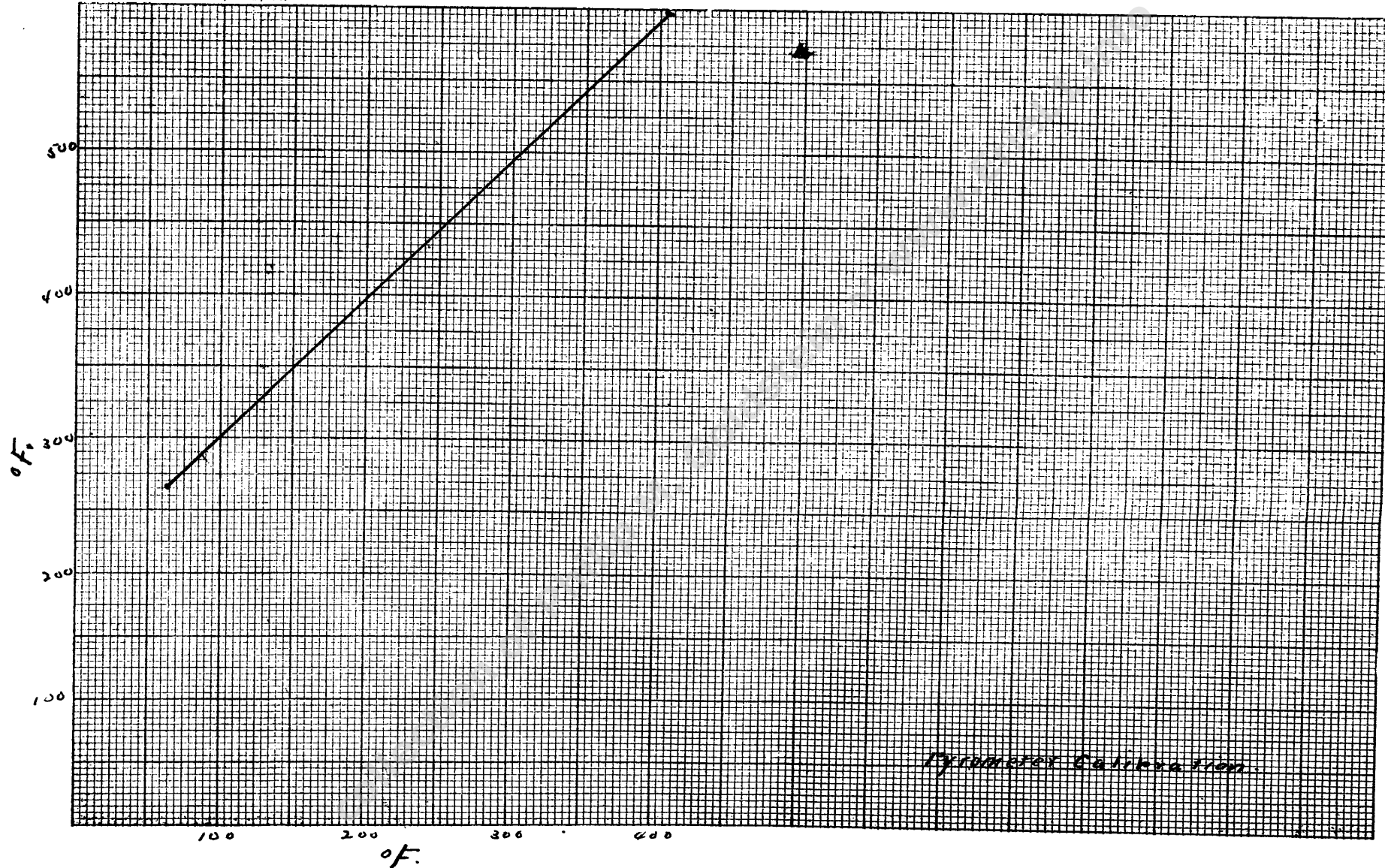


Calibration Curves
Indicator Springs

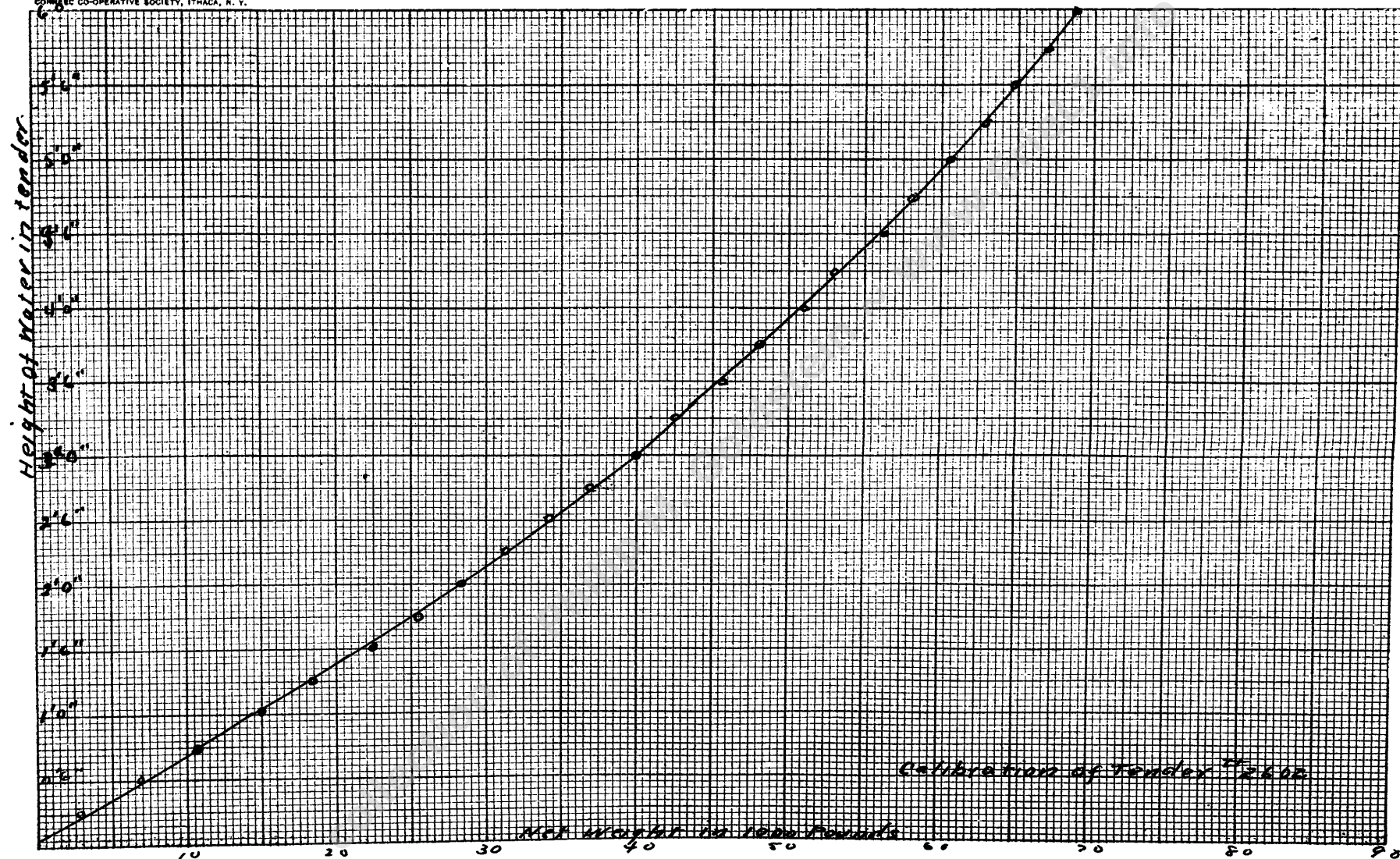
Left Side.

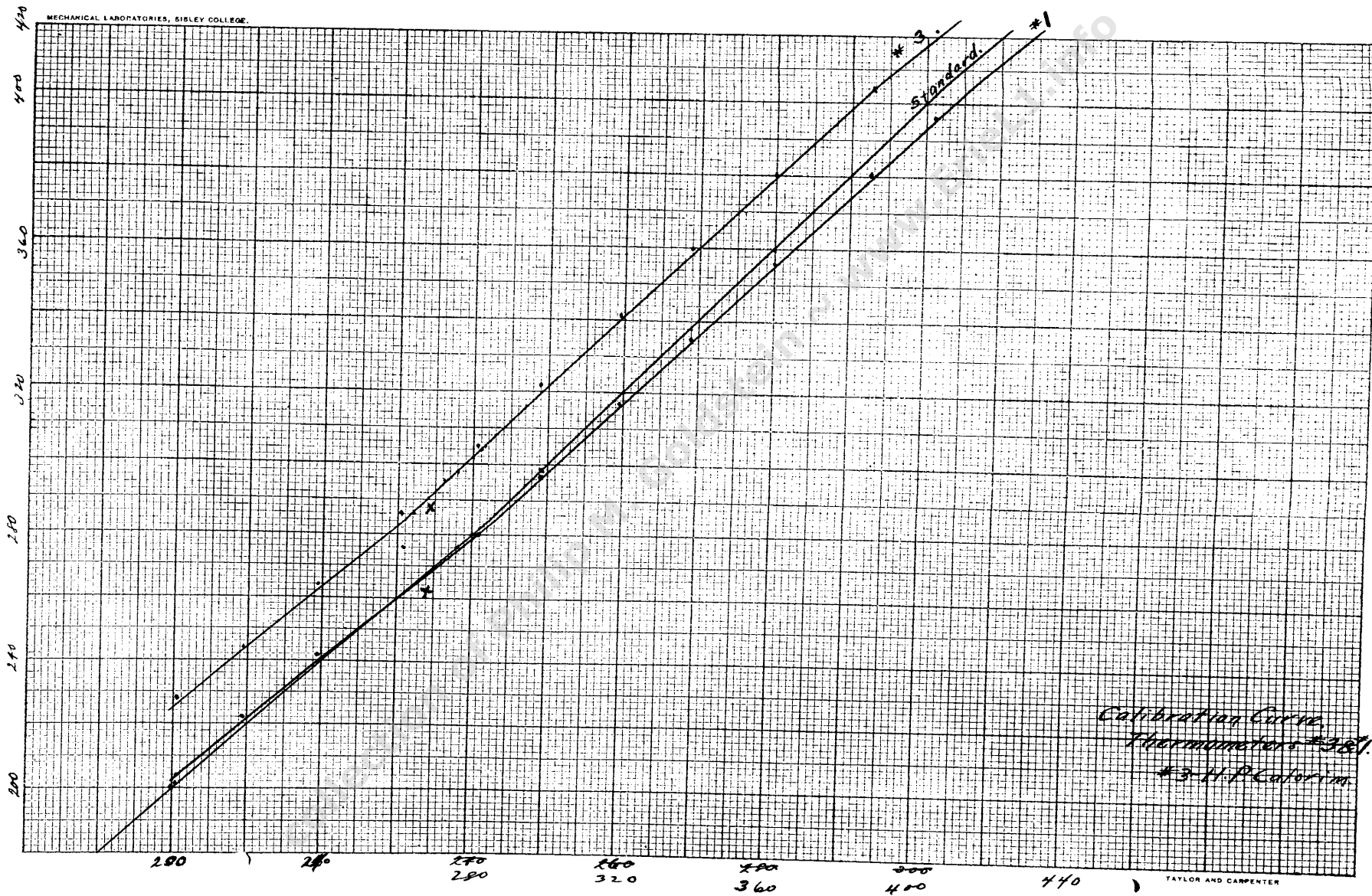
H.P. 150 $\frac{1}{2}$ " L.P. 60 $\frac{1}{2}$ "

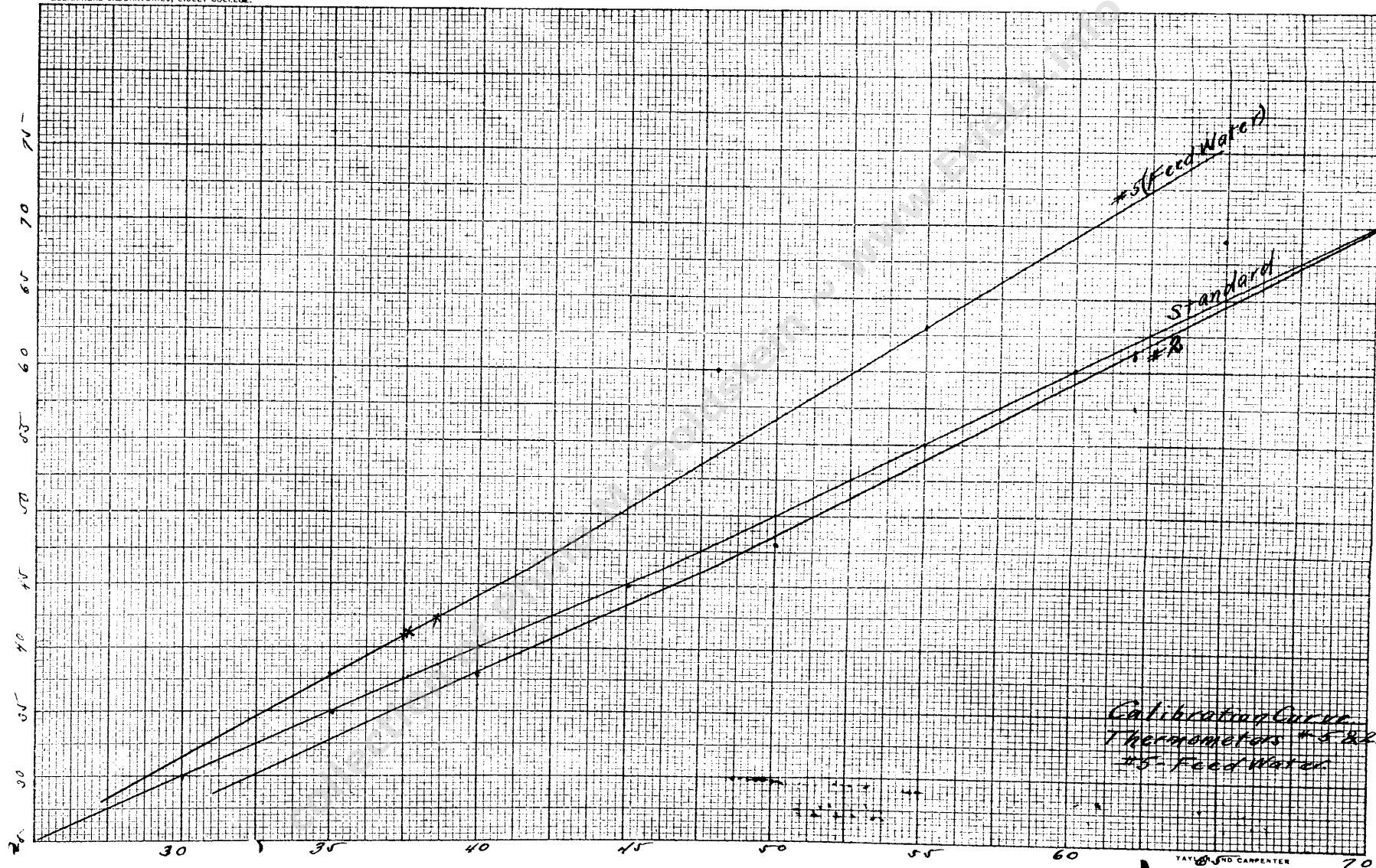


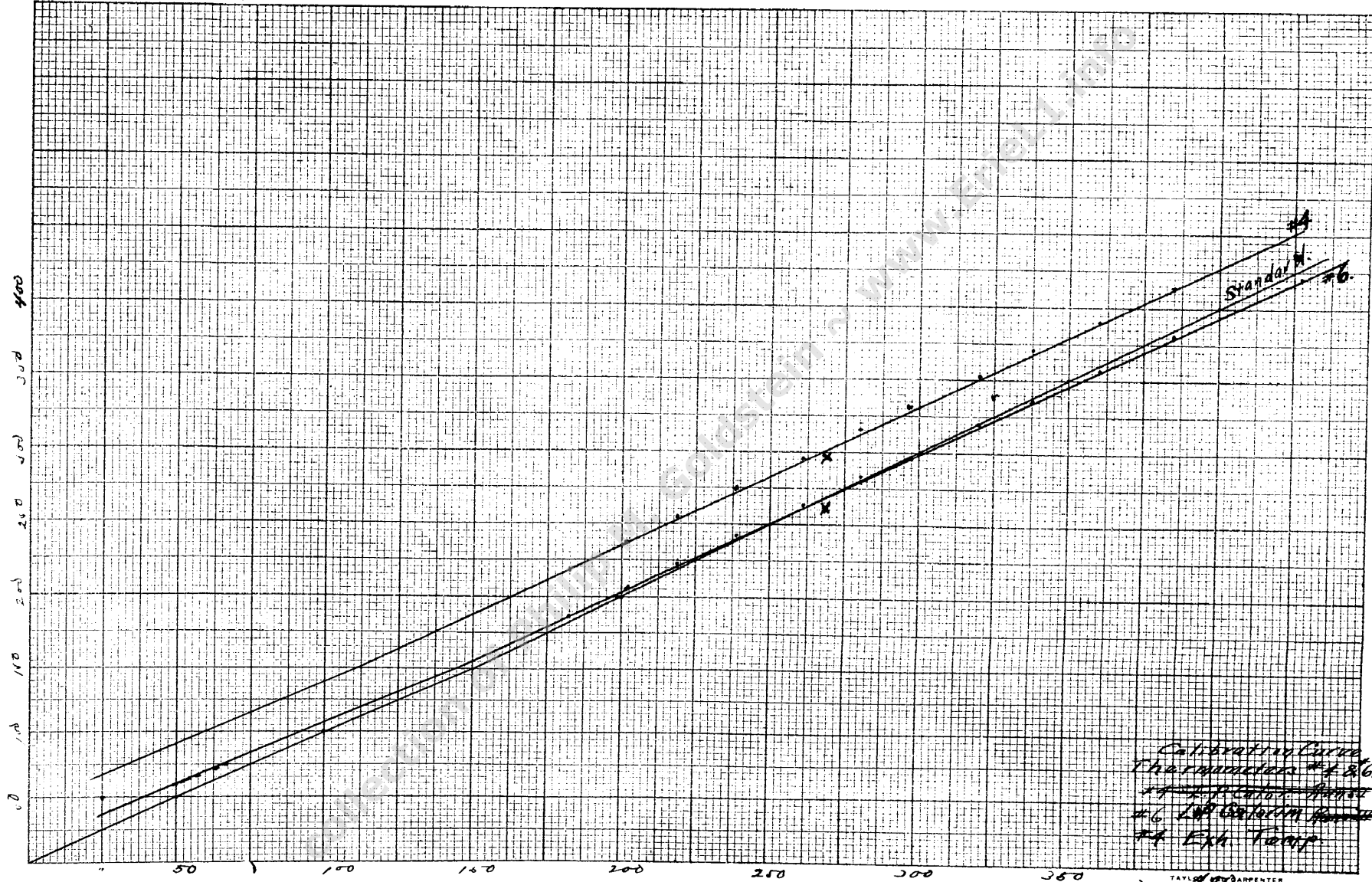


Pyrometer Calibration

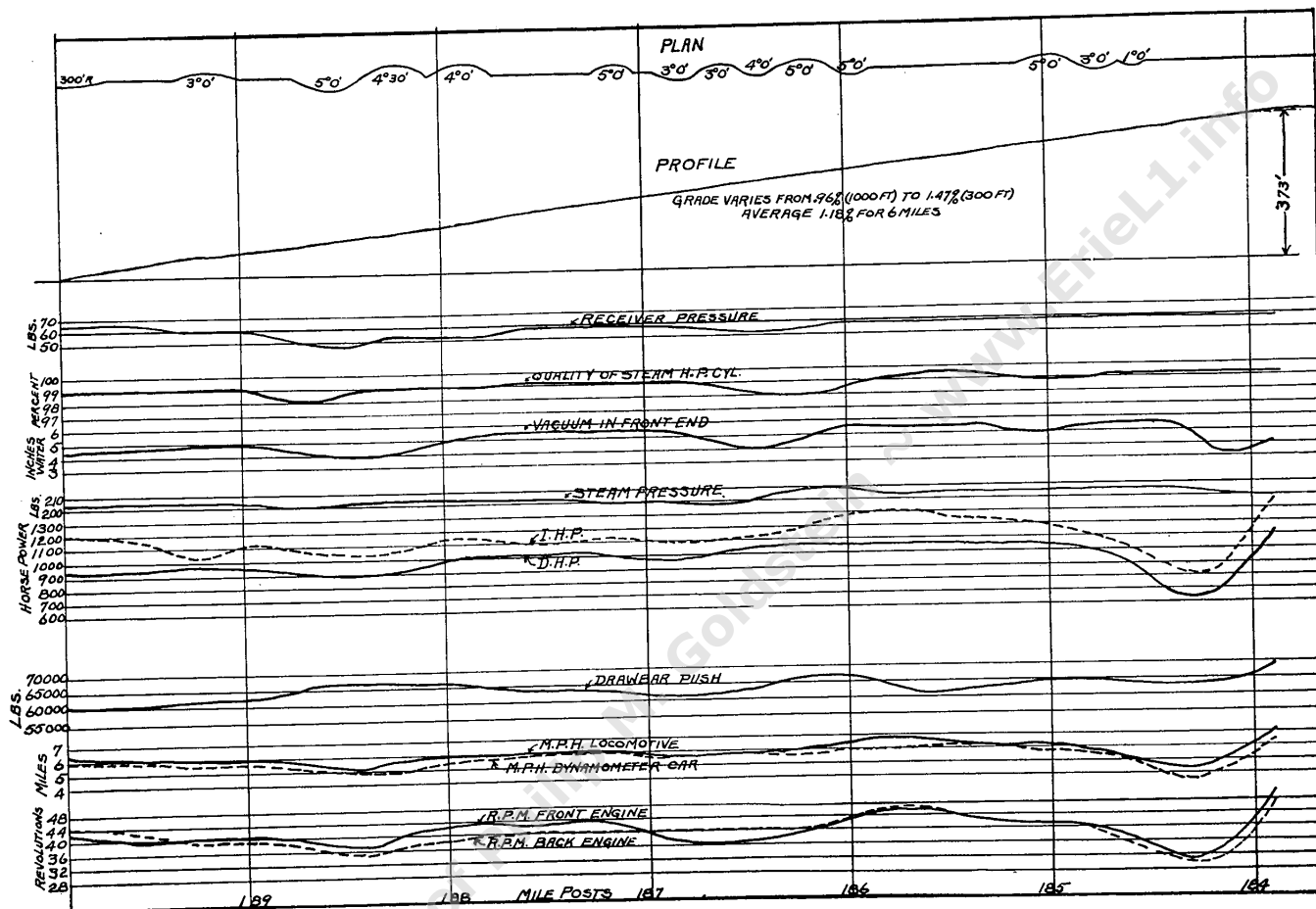








Calibration Curves
 Thermometers #4 & #6.
~~#4 & #6 Calorimeter 811~~
 #6 100 Calorimeter 811
 #4 Exh Temp



Log of Run 1.

